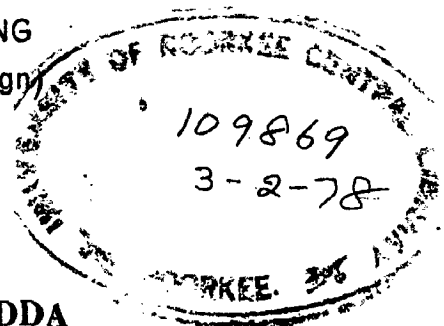


# STUDIES ON THE SUPERIMPOSITION OF AXIAL FLOW ON SWIRLING FLOW IN HORIZONTAL TUBES

A DISSERTATION  
submitted in partial fulfilment  
of the requirements for the award of the Degree  
of  
MASTER OF ENGINEERING  
in  
CHEMICAL ENGINEERING  
(Equipments & Plant Design)

By  
VINAI RAJ SINGH HUDDA



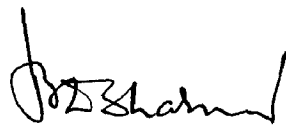
DEPARTMENT OF CHEMICAL ENGINEERING  
UNIVERSITY OF ROORKEE  
ROORKEE (INDIA)  
1976

C E R T I F I C A T E

Certified that the thesis entitled "STUDIES ON THE SUPERIMPOSITION OF AXIAL FLOW ON THE SWIRLING FLOW IN HORIZONTAL TUBES" which is being submitted by Sri Vinod Raj Singh Bhatta in partial fulfilment of the requirements for the award of the Degree of MASTER OF ENGINEERING in CHEMICAL ENGINEERING (Equipment and Plant Design) of University of Roorkee, Roorkee is a record of candidate's own work carried out by him under the supervision and guidance of the undersigned. The matter embodied in this thesis, has not been submitted for the award of any other Degree or Diploma.

This is further certified that he has worked for a period of seven months for preparing this thesis at this University.

October 18, 1976.



(D. D. Sharma)

Reader

Chemical Engineering Department  
University of Roorkee  
Roorkee.

## ABSTRACT

The present work deals with the swirling flow in horizontal tubes achieved by using swirl chamber, where the liquid is forced into the swirl chamber, through two tangential orifices located diagonally opposite to each other. Earlier workers have found that the flow reversal in swirling flow can be removed by giving air input at the axis of the tube. The same technique has been used in the present study also. Uptill now no study seems to have been carried out on the superimposition of axial flows on the tangential flows.

Therefore axial flow has been superimposed on the tangential flow and its effects on the air core diameter and length have been studied along with the air input rates. The studies have been carried out in two porous tubes of diameter 3.810 cm and 5.710 cm. It has been found within the range of the experimental conditions used, it is possible to superimpose axial flows on tangential flows without much disturbing the stability of the aircore. It has been found that in such cases much higher input rates are to be taken while superimposing axial flows as compared to air rates used without any axial superimposition.

## ACKNOWLEDGEMENT

I express my great sense of gratitude and indebtedness to Dr. D.D. Sharma, Reader, Department of Chemical Engineering, University of Roorkee, Roorkee who provided inspirational guidance, great encouragement, whole hearted cooperation and painstaking efforts at all stages of this work.

I also extend my hearty thanks to Dr. P.S. Panesar for his keen interest in the problem, fruitful discussions and suggestions at various stages of the work.

I am grateful to Dr. N. Gopal Krishna Professor and Head, Chemical Engineering Department for providing various facilities, help and constant encouragement.

Special thanks are due to Sri Achok Kumar, student, M.E. Chemical Engineering, 1st year, the staff of laboratory, fabrication section and the Departmental stores for their helping attitude throughout this work and

All those who contributed their own bit in completing this work.

V.R.S. Mada

(V.R.S. Mada)

## CONTENTS

	<b>Page</b>
<b>CERTIFICATE</b>	
<b>ABSTRACT</b>	
<b>ACKNOWLEDGEMENTS</b>	
<b>CONTENTS</b>	
<b>NOMENCLATURE</b>	
<b>CHAPTER I INTRODUCTION AND LITERATURE REVIEW</b>	1-16
<b>1.1 Introduction</b>	
<b>1.2 Literature Review</b>	
<b>1.3 Works Carried out in the Department</b>	
<b>CHAPTER II EXPERIMENTAL SET UP</b>	17, 18
<b>CHAPTER III OBSERVATIONS</b>	19-40
<b>CHAPTER IV DISCUSSION OF RESULTS</b>	49-52
<b>CONCLUSIONS AND RECOMMENDATIONS</b>	5
<b>CHAPTER V BIBLIOGRAPHY</b>	53, 54

-----

## NOUENCLATURE

$D_a$	Average diameter of air-core	cm
$D_1$	Internal diameter of tangential entry	cm
$D_0$	Internal diameter of swirl chamber	cm
$D_T$	Inside diameter of tube (test section)	cm
$G$	Gravitation Acceleration	cm/sec <sup>2</sup>
$G_c$	Gravitation (Newtons) conversion factor	
$L$	Length of stable air core	cm
$\Delta P$	Pressure drop of liquid measured at tangential inlets	Kg/cm <sup>2</sup>
$R$	Radius at any point	cm
$R_0$	Tube radius test section	cm
$V_1$	Velocity of liquid at tangential inlets	cm/sec.
$V_3$	Tangential component of the velocity at any point	cm/sec.
$Z$	Axial distance down main tube referred from inlet plane	cm
$\rho$	Density of liquid	gm/cm <sup>3</sup>
$\nu$	Kinematic viscosity	cm <sup>2</sup> /sec
$C$	Circulation constant	

**CHAPTER I**

**INTRODUCTION AND LITERATURE REVIEW**

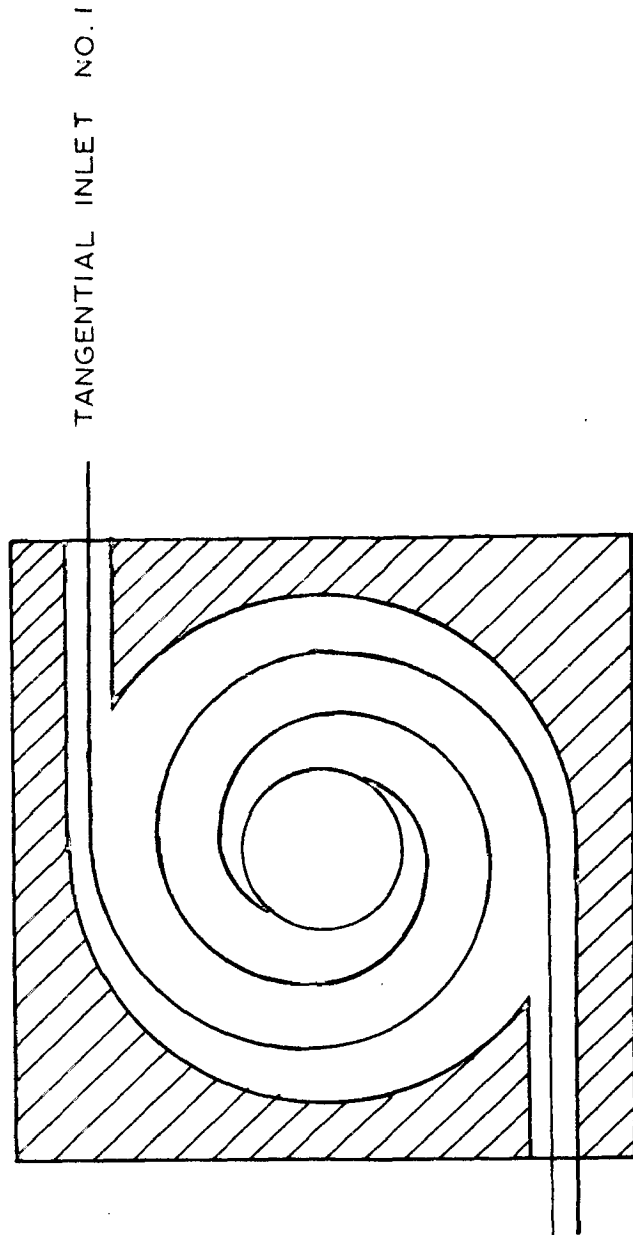
## 1.1 INTRODUCTION

The flow characteristics of rotating systems are of great theoretical and practical importance and have been a subject of active interest for long time. Review of literature reveals that a paper was published(1) as early as 1880 on the gravitational oscillations of rotating water by Kelvin. The flow characteristics of rotating systems are complex and in addition most of the systems reveal novel and unexpected flow characteristics when put to a complete stability analysis.

Considerable work has been done in swirling flow through atomizers, fluid flow between rotating cylinders and in curved channels. Attention of various workers have been drawn on compressible swirling flow in tubes since 1950, however, relatively little work is available in literature regarding swirling incompressible flow in long tubes. The swirling flow inside long tubes can be generated by any one of the following methods:

- (1) By rotating the tube through which the fluid is pumped.
- (2) By introducing fluid into the tube with the help of guided vanes.
- (3) By inserting twisted tape in the tube through which the fluid is pumped.
- (4) By forcing liquid into the tube through tangential inlets at the inner surface of the tube.





TANGENTIAL INLET  
NO 2

TANGENTIAL INLET NO. 1

FIG. 1

PRODUCTION OF SWIRLING FLOW BY  
TANGENTIAL INJECTION OF LIQUID

### GENERAL PRINCIPLE :

Principle of swirl flow generation can be understood by the help of fig. 1. showing formation of swirling flow by two tangential inlets. As the liquid enters through a tangential inlet No. 1 it takes a swirling motion and as it meets the liquid from the tangential inlet No. 2 it is pushed towards the centre of the tube. Thus a swirling motion is established. This swirling motion can be of two types

- (1) Forced vortex flow in which angular velocity is constant and
- (2) Free vortex motion in which the moment of momentum is radially conserved such that the product of the tangential velocity component and radius at any point within the main body of the liquid is constant i.e.

$$V_T \times R = \text{---} \quad \text{---} \quad = (1.1)$$

where  $\text{---}$  is generally referred to as circulation constant.

Swirling flow produced in tubes is generally more of free vortex in nature. In this type of flow we have already seen that the moment of the momentum is radially conserved i.e. product of tangential velocity component and radius at any point within the main body of the fluid remains constant. This will predict creation of vacuum at the centre and hence reversal of axial flow is predicted even at lower strength of swirl.

From equation 1.1 it is apparent that as  $R$  approaches zero  $V_T$  approaches infinite which is hydrodynamically impossible. The axial velocity component has been reported to cavitate near the central axis if the swirl strength is low. At higher values of swirl strength a complete reversal of axial component was reported still higher swirl strength give rise to vacuum of such a degree that a central air-core is produced if the system is able to suck air at the axis of swirl from some source. If this air passage is sealed the liquid itself vapourises to give rise to vapour core.

## 1.2 LITERATURE REVIEW

Talbot(2) studied the swirling flow of water in a tube and employed a long unperforated tube in which swirl was induced by the rotation of tube situated at some distance from the entry. The swirl strength was insufficient to produce flow reversal but a dimpling of the axial velocity profile at the centre of the tube was reported.

Binnie and Tearo(3) carried out experiments on the flow of swirling water through a pressure nozzle and recorded observations when the swirling water was discharged downwards under pressure through a large perspex conical nozzle. It was discovered that a boundary layer of forced vortex motion existed around the free surface of the aircore. The results were confirmed by measurement of tangential and

axial velocities close to the free surface. Further when the swirl was sufficiently strong compared with the supply pressure the axial component of the velocity was reversed in the upper part of the nozzle close to the forced vortex core.

Binnie(4) investigated the nature of the flow patterns when swirling water was passed through a long transparent horizontal straight tube. A portion of the tube (perspex tube of I. D. 2.0") was rotated and water was admitted by gravity into the tube through a large number of holes. The flow through the apparatus was controlled by altering the heights of the constant level tank, from which the water was fed. The maximum speed was about 40 revolutions per minute. Coloured liquid was injected and following three regimes were observed :-

- (1) Down stream over entire cross section.
- (2) Upstream near the axis and down stream near the tube wall.
- (3) Down stream near the axis and the wall and upstream in the intermediate region.

Nuttal(5) carried out the study of swirling fluid flow in circular pipe. His experimental apparatus consisted of 2.875" I. D. 63" long perspex pipe set vertically and mounted by cylindrical tank containing a ring of guided vanes which produced the swirl (free vortex). The rate of discharge was controlled by throttling the lower end of the pipe by using several throttling devices. Dye injection technique was used for observing fluid path lines

No quantitative velocity measurements were made in his study. His observations were as follows:

- (1) At low rates of swirl the axial velocity at the pipe centre was less than the expected maximum velocity.
- (2) As average axial velocity increased, centre line velocity decreased to a point where it reversed in direction.
- (3) Finally upon further increase of the average vertical velocity, centre line axial velocity was positive at the centre but negative in a narrow region between the axis and the wall of the pipe.

His qualitative observations confirmed that flow reversal occurs in the centre of the pipe for some ratio of swirl and discharge. Similar observations were observed by Binns (4) and were called regions I, II and III respectively. Although Binns's method of producing swirl was different than that of Nuttal's.

Pleso et al (6) recorded investigations on the behaviour of the air flowing through large spray drier by making use of tracer technique and found the existence of the swirl flow pattern in spray drier. While studying the flow pattern of the spray drier in a model they found a region within their experimental equipment

where a core of velocity opposite in direction to the primary flow existed. A transparent 1/80th scale model was used in which the fluid used was water and the flow entered the model through 12 ports placed at appropriate angles to generate the swirl. Coloured solutions were injected into the model through these inlets.

The experiments conducted as above revealed the presence of a zone of reversed flow and they could observe a region of downward vertical velocity located between an upward central core and an upward velocity region adjacent to the wall. Values were not ascribed to the velocities but it was described as a "Region III type of flow".

Kwath and Sonju (7) studied the decay of a liquid swirl which was induced by twisted metal strips along the centre line of a tube in an one inch pipe. It was observed that the swirl decay to about 10-20% of its initial intensity in a distance of about 50 pipe diameters the decay being more rapid at small than at large Reynolds numbers. In this work the swirl was not sufficiently strong to produce flow reversal. The theoretical swirl velocity distribution agreed qualitatively with experimental measurements at distance less than 20 diameter down stream from the outlet of the swirl inducer, but deviated from the experimental results further down stream.

Smithberg and Landis (8) while studying the convection heat transfer characteristics in tubes with twisted tape swirl generators also studied the velocity distribution and made the following conclusions.

- (1) The axial velocity is nearly constant over most of the cross section while the inplane component appears to be tangential and increase linearly with radius.
- (2) The velocity field is helicoidal and corresponds to a forced vortex in the core superimposed on an essentially uniform axial flow.

The author claims that a helicoidal flow with a uniform axial velocity with its total velocity vector tangent at every point to an helix defined by the pitch of the twisted strip and the radial distance from the tube centre line will result in a forced vortex motion in the tube cross section. This predicted vortex pattern was found to be in excellent agreement with the experimental results.

Broman et al (9) studied the swirling incompressible flow of water in a tube held vertically with the swirl induced by tangential injection of the water into the test section from bottom of the tube. The inlet section consisted of 2 tangential inlet pipes which were parallel and  $100^\circ$  apart and flush with the backplate. Inlet diameters were equal to  $D/4$ . The tube diameter

2" and 2.5" I.D. were used for the 100" long test section. Water was pumped upwards through the vertical tubes. Lucite plastic tube were used for the test section so that visual observations could be made. In this investigation a study of pressure and velocity profile was made in an attempt to elucidate some of the factors governing the phenomenon of reversed flow. Profiles were studied at four lengths of pipe and for Reynolds numbers ranging from 5,000 to 25,000 with the help of two pressure probes (used separately to measure total and static pressures respectively) Reynolds numbers were measured based on the diameter of the tube and average vertical velocity in the tube. Their conclusions regarding velocity profiles were as under:

- (1) Swirling flow produced by injecting water at constant rate and tangentially into a cylinder through two ports situated diametrically opposite to each other is found to be steady and approximately cylindrical and coaxial to the tube. Injection of water through a single port gave a spiral core and destroyed the cylindrical symmetry.
- (2) Except in narrow regions near inlets and outlets of the cylinder radial velocities were always small compared to the tangential and axial velocities.



- (3) The tangential velocity profile across a radius showed the velocity to increase from zero at the centre to a maximum at a radius usually less than half the tube radius to keep steady for the further distance and then to fall to zero again at the wall indicating that the flow was essentially forced vortex in nature.
- (4) The mean tangential velocity, averaged across a diameter steady decreased down stream from the inlet.
- (5) The axial referred to as vertical in their experiments showed the most interesting variations. For an overall upward flow along the tube they observed
- (a) At low rates of flow the vertical velocity was upward throughout the cylinder
  - (b) At higher rates of flow there was reversal of flow at the axis of the tube so that water flowed downwards at the centre, reversed direction near the inlet and upward near the wall.

King et al (10) in their investigations in swirling incompressible tube flow took a system similar to Dreesen et al. Their studies was carried out in 2" I.D. test section of plain glass tube 10' long held horizontally. The swirl was introduced by injection of

the total fluid stream through two symmetric tangential inlets  $1/2''$  diameter perpendicular to the tube with equal flow rates to ensure axisymmetric flow. They also obtained static pressure and velocity profile by using probe technique. These profiles were determined for test section axial flow Reynolds numbers of 10,000, 15,000, 20,000 and 25,000. They concluded the following observations regarding the velocity profiles.

- (1) The flow rate could be roughly divided into free and forced vortex regions.
- (2) A region of reverse axial flow was noted in the tube centre, the radius of this region decreasing with increasing  $Z/R_0$  to zero ( $Z$  = axial distance from the inlet of the main tube,  $R_0$  is tube radius of test section).
- (3) Also curves of the tangential velocity/inlet velocity versus  $Z/R_0$  were developed as a form of representation of swirl decay.

Pressure drop studies in swirling flow created by guided vanes, by rotation of the tube apparently has not been reported in the literature.

Smithberg et al (8) in their studies have concluded that the friction losses may be predicted from the combined effect of the axial and the tangential

boundary layer flows coupled with an additional "Vortex mixing effect". They have also proposed an equation for predicting the total vortex mixing loss with satisfactory accuracy for tube with twisted tape turbulence generators. The range is applicable for Reynolds number in the range from 2,000 to 100,000.

Lopina et al (11) have made studies for the pressure drop in tape generated swirl flow of single phase water. The studies were carried out for electrically heated tubular test sections with tight fitting full length tapes with twist rotation from 2.5 to 9.2. They have also reviewed the work of earlier investigators on this subject for various systems like air, other gases, water, liquid metals in generated swirl flow studies. These workers have suggested a simpler equation for predicting the isothermal friction factor for swirl flow. They also observed that the difference between the isothermal and heated friction factors for the swirl flow data was substantially less than the corresponding difference for the empty tube.

This investigation was carried out at the same time as that of Thorson and Landle who studied the friction; and heat transfer characteristics in turbulent swirl flow subjected to large transverse temperature gradient.

Bergles et al (12) carried out studies in rough tube with tape generated swirl flow. Pressure drop data

for a variety of tubular test sections with low pressure water system was taken and also various combinations of the tube roughness and swirl flow were tried and utilized. They have proposed an expression for the isothermal swirl flow friction factor.

Droem et al (9) recorded observations regarding the pressure profiles while studying the swirling incompressible flow of water in a tube held vertically with the swirl induced by tangential injection of water into the test section from bottom of the tube. Their conclusions were:

- (1) Profiles of the static pressure along any radius of the tube always showed a minimum at the centre with a more or less steady increase towards the wall.
- (2) The static pressure profile as well as patterns of flow established by the axial velocities were primarily controlled by the variations of tangential velocity along the radius of tube and its decay along the axial direction. Boundary layer growth appears to offer an explanation for the doubly reversed flow described as above.
- (3) The static pressure measured at the wall decreased steadily down stream from the inlet.

King et al (10) who conducted the studies in swirling incompressible horizontal tube flow, with the swirl induced by tangential injection of the water into the test section made observations regarding pressure profiles and concluded that the static pressure  $Z/R_0$  increases monotonically from the centre line to the wall with increasing  $Z/R_0$ , the magnitude of this gradient decreases as the gradient producing swirl decays. In addition it is particularly interesting to note that at, and near, the centre line the axial pressure gradient is positive for considerable distance from the tube

### 1.3 WORK CARRIED OUT IN OUR DEPARTMENT

Considerable studies have also been conducted in our department in the field of swirling flow in horizontal and vertical tubes.

Studies in swirling flow in vertical tubes have been conducted by Shri Goel (16) and Shri Chandra (15). Mr. Goel (16) produced the swirling flow by injecting water through the tangential inlets at the bottom of the tube. He employed a 3.80 cm internal diameter and about 2.0 meters long perspex tube for his studies. In his studies he has confirmed the existence of the flow reversal and also shown that the flow reversal can be removed by injecting air at the axis of the vertical tube. Further

Sri Gool has carried out a photographic study for the above phenomenon of flow reversal and its removal. Flow pattern studies were done with the help of dye injection technique. Effects of various geometrical and flow dynamic parameters on pressure drop in such a flow have been studied. Theoretical method has been proposed to compute the pressure drop in this type of flow. His studies contains the production of swirling flow by using a tangential entry block as well as by using a swirl chamber.

Mr. Chandra (15) has carried out the swirling flow studies in vertical tubes by injecting liquid through tangential inlets from the top of the tube and allowing the water to flow downwards. The tubes employed by Sri Chandra were of perspex ranging from 1.80 to 5.60 cms in internal diameter and 130.- cms in length. The upper end of the tube was screwed in the tangential entry head. Shri Chandra has also confirmed the existence of flow reversal and its removal by the technique used by Sharma (15). In swirling flow the pressure drop across the entry head and the tube through which the liquid is flowing is the summation of the friction is due to sudden expansion of cross section of the pipe, the radial pressure drop at the entry head, the pressure drop due to swirling flow inside the tube and the pressure drop due to friction in the entry pipes. As the various parameters involved were not easily amenable for measurement and in view of the difficulty he has proposed the following dimensionless

correlation

$$Eu = (Ro)^{-0.41} (Fr)^{0.01} (D_1/D_T)^{-0.03} (L/D_T)^{-0.85}$$

Sharma et al (13,193) have developed a new method for generating swirling flow by use of separate swirl chamber and exposing the axis of the swirl to atmosphere or injecting air thereby breaking the flow reversal. In the studies carried out in our laboratories in this depth they have studied the incompressible rotating liquids without flow reversal in horizontal tubes of four different diameters and lengths for 2.0 metres to 4.0 metres approximately. They have carried out experiments and developed various correlations for predicting pressure drop, air core diameter and length as stated below.

Sharma has also revealed the existence of flow reversal in such systems and its removal with the technique proposed and confirmation of the same work with the help of dye injection and photographic studies.

**CORRELATION FOR PRESSURE DROP:**

$$\frac{\Delta P_{GC}}{V_1^2} = 2.57 \times 10^{-2} \left[ \frac{D_T V_1}{\nu} \right]^{-0.15} \left[ \frac{V_1}{G D_T} \right]^{-0.065} \left[ \frac{D_G}{D_T} \right]^{4.18} \left[ \frac{D_T}{D_1} \right]^{2.6}$$

## CORRELATION FOR DIAMETER OF CORE

$$\frac{D_c}{D_T} = 0.94 \left[ \frac{D_c}{D_T} \right]^{-0.50} \left[ \frac{D_c}{D_T} \right]^{0.16}$$

## CORRELATION FOR LENGTH OF CORE

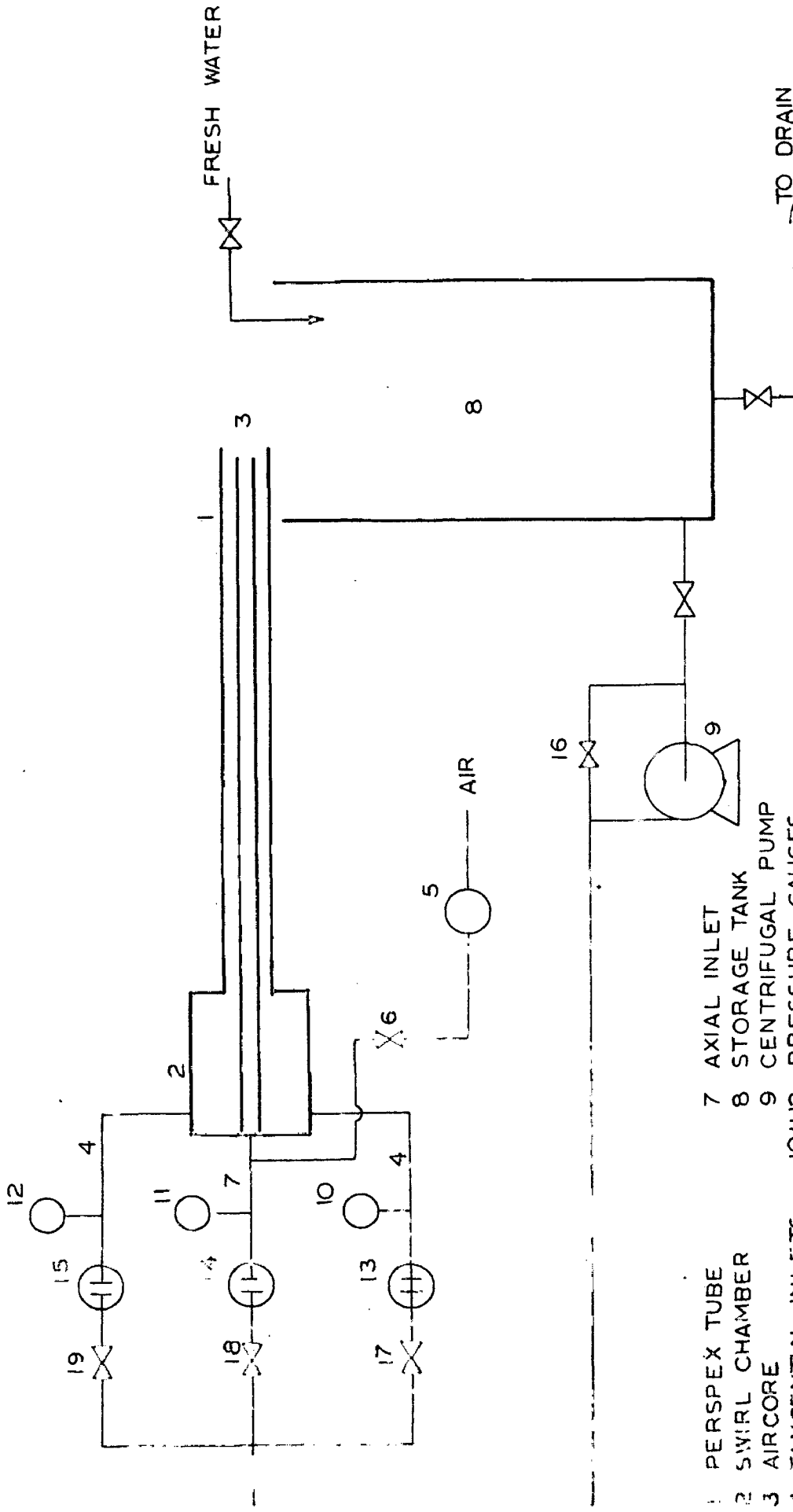
$$\frac{L}{D_T} = 2.72 \left[ \frac{D_T V_1}{v} \right]^{0.41} \left[ \frac{V_1}{G D_T} \right]^{0.17} \left[ \frac{D_c}{D_T} \right]^{1.05} \left[ \frac{D_c}{D_T} \right]^{1.58}$$

They have also mentioned that it is expected that probably the air core diameter and length of swirling flow in horizontal tubes will increase by super-imposing some horizontal axial flow on the tangential flow without much disturbing the stability of the aircore in the tangential flow. No work seems to have been done in this respect in the study of swirling flow produced by tangential inlets in horizontal tubes. Therefore, it was proposed to study the effect of super-imposition of axial flow on the stability of swirling flow in horizontal tubes and its effect on the stability so that swirling flow systems can be better utilized in industries.



CHAPTER II

EXPERIMENTAL SET UP



- 1 PERSPEX TUBE
- 2 SWIRL CHAMBER
- 3 AIR CORE
- 4 TANGENTIAL INLETS
- 5 WET GAS METER
- 6 AIR INLET
- 7 AXIAL INLET
- 8 STORAGE TANK
- 9 CENTRIFUGAL PUMP
- 10 11 12 PRESSURE GAUGES
- 13 14 15 ORIFICE METERS
- 16 17 18 19 GLOBE VALVES



EXPERIMENTAL SET UP

## CHAPTER XX

### ROTATIONAL FLOW

The proposed experimental set up is shown in the Fig. (2) attached herewith. It consists of a long porous tube (1) Tubes of different inside diameter can be used as required. Tubes of different lengths are used. A separate mild steel swirl chamber (2) is fitted to one end of the tube by stuffing box arrangement. The swirl chamber consists of one axial (7) and two symmetrically located tangential inlets (4) for feeding liquid to the swirl chamber. An air inlet (6) is provided for sucking or pushing air at the centre of axis of swirl. When the tangential inlet velocity is high enough to give rise to a free vortex type of motion a central air core is carried to the tube is obtained to a long distance inside the tube by allowing the air at the axis, measured by a calibrated wet gas meter (5). Water from a storage tank (8) is drawn by J.H.P. centrifugal pump (9) and fed into both the tangential inlets at equal pressures, to obtain a stabilised swirling flow in the swirl chamber. The inlet pressures are recorded by calibrated pressure gauges (10, 11, 12). The flow of water was measured by calibrated orificemeters (13, 14, 15) and also verified by direct measurements. Globe valves (16, 17, 18, 19) are used for controlling flow of water. The experimental set up is shown in Fig. 2 for the studies of superimposition

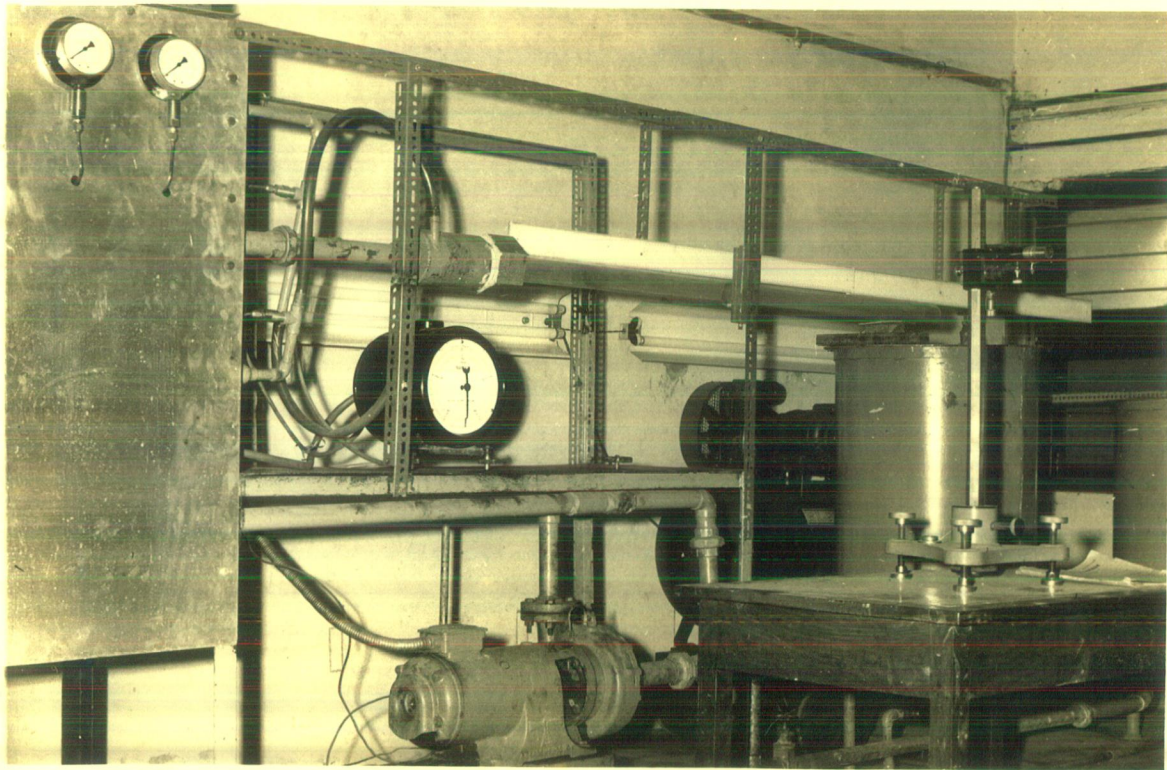
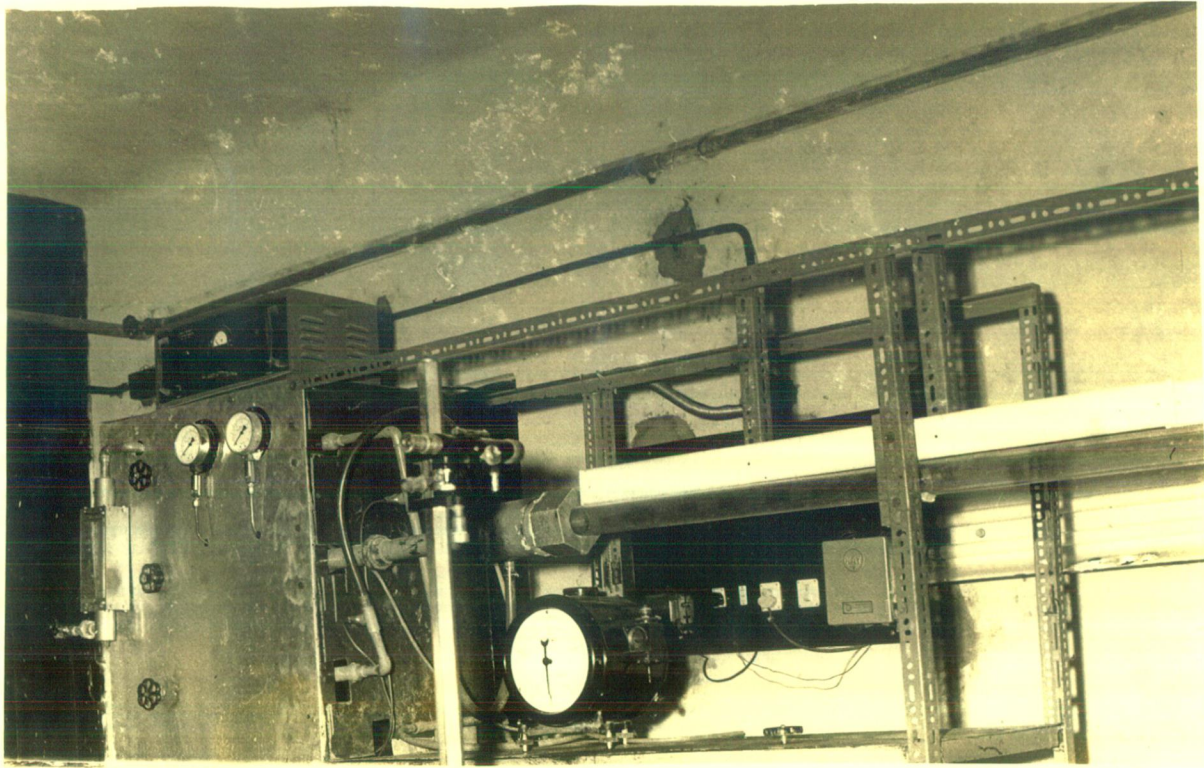


PLATE N<sup>o</sup> 1 AND 2

EXPERIMENTAL SET UP

of axial flow on tangential flow. A photograph of the same is shown in Plate No.1.

#### AIR-CORE MEASUREMENT

##### (a) Diameter:

The air-core diameter was measured with the help of a calibrated cathetometer having an eye piece calibrated for distance and refraction. The diameter of air-core was measured along the entire length of the tube. The average diameter was then computed. Care should always be taken to maintain reasonably straight and uniform air core throughout the length of the tube.

##### (b) Length:

A scale graduated in centimeters is kept fixed along the entire length of the porous tube and the length of the air-core can be directly noted on this scale.

#### MEASUREMENT OF AIR RATE:

The quantity of air is to be measured with the help of a calibrated wet gas meter. The inlet pressure of air is to be recorded by a manometer.

CHAPTER III  
OBSERVATIONS

Table No. 1

Tube Diameter

= 3.81 cms, Tangential flow rate = 5700 LPH

Air flow rate

= 280 LPH, Correction factor for refraction=0.87

Sr. No.	Distance along the length of tube (Cms.)	Corrected air core diameter (in cms) for superimposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	2.435	2.580	2.364	2.297	2.373	2.306
2	10	2.371	2.673	2.445	2.388	2.343	2.223
3	20	2.351	2.551	2.565	2.413	2.300	2.325
4	30	2.507	2.518	2.377	2.372	2.375	2.279
5	40	2.516	2.462	2.355	2.326	2.262	2.288
6	50	2.480	2.466	2.323	2.311	2.366	2.306
7	60	2.459	2.419	2.393	2.284	2.268	2.227
8	70	2.393	2.359	2.292	2.271	2.339	2.245
9	80	2.352	2.314	2.350	2.194	2.253	2.185
10	90	2.245	2.266	2.187	2.122	2.172	2.219
11	100	2.198	2.137	2.153	2.038	2.093	1.968
12	110	2.158	2.248	2.133	1.934	2.114	1.956
13	120	2.137	2.144	2.114	1.888	2.003	1.917
14	130	2.074	2.057	1.960	1.884	2.060	1.844
15	140	2.057	2.067	1.953	1.949	2.058	1.890
16	150	2.005	1.843	1.836	1.815	1.935	1.871
17	160	1.937	1.897	1.844	1.827	1.764	1.740

Table 2

Tube Diameter = 3.01 cm Tangential flow rate = 5700 LPM

Air flow rate = 200 LPM • Correction factor for extraction = 0.87

Sr. No.	Distance along the length of the tube (cm)	Corrected air core diameter (in cm) for captured axial flow (in LPM)					
		0	100	200	300	400	500
1	0	2.473	2.463	2.373	2.406	2.327	2.269
2	10	2.423	2.563	2.357	2.464	2.309	2.232
3	20	2.393	2.532	2.673	2.422	2.348	2.263
4	30	2.376	2.513	2.527	2.429	2.311	2.333
5	40	2.349	2.455	2.395	2.332	2.321	2.275
6	50	2.333	2.436	2.232	2.315	2.294	2.238
7	60	2.311	2.435	2.363	2.263	2.317	2.307
8	70	2.305	2.455	2.215	2.302	2.253	2.305
9	80	2.295	2.389	2.275	2.067	2.256	2.152
10	90	2.266	2.272	2.262	2.251	2.227	2.265
11	100	2.281	2.267	2.262	2.232	1.933	2.091
12	110	2.163	2.212	2.225	2.101	2.013	1.953
13	120	2.115	2.272	2.255	2.045	2.023	1.950
14	130	2.692	2.033	2.205	2.070	1.888	1.924
15	140	2.077	2.696	2.220	1.045	1.925	1.924
16	150	2.003	1.837	1.927	2.001	1.923	1.805
17	160	1.934	1.825	1.920	2.001	1.762	1.744



Table No. 1

$r = 3.81$  cm      Tangential flow rate = 6700 LPH  
 $z_0 = 650$  LPH      Correction factor for porosity = 0.07

z along length of tube z)	Corrected air core diameter (in cm) for capillary imposed axial flow (in LPH)					
	0	100	200	300	400	500
2.576	2.366	2.376	2.362	2.326	2.35	
2.572	2.506	2.460	2.445	2.429	2.36	
2.580	2.472	2.553	2.452	2.423	2.23	
2.529	2.423	2.426	2.399	2.338	2.39	
2.523	2.425	2.620	2.382	2.352	2.36	
2.383	2.335	2.322	2.362	2.369	2.37	
2.436	2.433	2.370	2.352	2.327	2.31	
2.433	2.370	2.393	2.333	2.363	2.28	
2.373	2.369	2.293	2.299	2.239	2.26	
2.396	2.333	2.210	2.290	2.230	2.26	
2.236	2.239	2.253	2.237	2.269	2.25	
2.273	2.217	2.275	2.096	2.252	2.26	
2.093	2.213	2.202	2.065	2.035	2.01	
2.065	2.293	2.202	2.023	2.026	2.01	
2.002	2.272	2.202	2.099	1.970	1.91	
2.093	2.069	2.020	2.060	1.960	1.81	
2.069	2.071	2.053	2.053	2.057	2.01	

Table No. 4

Tube Diameter = 9.01 cm., Tangential flow rate = 4070 LPH

Air flow rate = 490 LPH Correction factor for restriction = 9.87

Sr. No.	Distance along the length of tube (Cm.)	Corrected air core diameter (in cm) for superimposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	2.555	2.556	2.449	2.416	2.228	2.218
2	10	2.517	2.581	2.553	2.458	2.340	2.244
3	20	2.449	2.524	2.459	2.564	2.289	2.295
4	30	2.485	2.448	2.459	2.595	2.576	2.221
5	40	2.470	2.423	2.423	2.359	2.51246	2.225
6	50	2.450	2.450	2.453	2.320	2.209	2.208
7	60	2.425	2.343	2.343	2.324	2.225	2.101
8	70	2.431	2.353	2.353	2.293	2.242	2.153
9	80	2.346	2.219	2.255	2.213	2.100	2.144
10	90	2.233	2.245	2.260	2.121	2.110	2.058
11	100	2.125	2.109	2.112	2.071	2.077	2.013
12	110	2.115	2.009	2.022	2.091	2.058	1.981
13	120	1.973	2.045	2.009	2.001	1.922	1.945
14	130	1.905	1.998	2.001	1.938	1.853	1.882
15	140	1.828	1.926	1.931	1.940	1.827	1.806
16	150	1.871	1.941	1.973	1.879	1.847	1.865
17	160	1.938	1.800	1.903	1.872	1.842	1.749

Table No. 5

Tube diameter = 9.81 mm ; Tangential flow rate = 4870 LPH  
 Air flow rate = 895 LPH ; Correction factor for reflection = 0.87

Sr. No.	Distance along the length of tube (mm)	Corrected air core diameter (in mm) for unimpeded axial flow in (LPH)					
		0	100	200	300	400	500
1	0	2.566	2.449	2.428	2.408	2.415	2.324
2	10	2.597	2.449	2.472	2.470	2.540	2.265
3	20	2.566	2.524	2.400	2.437	2.449	2.326
4	30	2.541	2.495	2.482	2.423	2.428	2.350
5	40	2.523	2.486	2.403	2.362	2.366	2.275
6	50	2.503	2.472	2.432	2.365	2.357	2.263
7	60	2.462	2.467	2.363	2.362	2.334	2.227
8	70	2.436	2.403	2.352	2.265	2.305	2.288
9	80	2.380	2.344	2.331	2.297	2.222	2.172
10	90	2.331	2.301	2.279	2.267	2.204	2.214
11	100	2.321	2.225	2.260	2.063	2.123	2.057
12	110	2.232	2.173	2.074	2.043	2.101	2.054
13	120	2.162	2.152	2.122	2.035	2.002	1.993
14	130	2.214	2.113	2.092	2.020	2.034	1.973
15	140	2.096	2.027	2.025	1.934	1.996	1.927
16	150	2.008	1.935	1.962	1.899	1.942	1.867
17	160	1.940	1.920	1.907	1.850	1.887	1.809

Table No. 6

Tube diameter = 3.81 cms, Tangential flow rate = 4070 LPH  
 Air flow rate = 1310 LPH ; Correction factor for refraction=0.87

Sr. No.	Distance along the length of tube (Cms).	Corrected air core diameter (in cms) for capillary-imposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	2.575	2.484	2.462	2.479	2.436	2.394
2	10	2.637	2.524	2.474	2.489	2.518	2.420
3	20	2.606	2.540	2.522	2.509	2.461	2.331
4	30	2.597	2.523	2.457	2.441	2.492	2.450
5	40	2.578	2.491	2.411	2.401	2.395	2.318
6	50	2.514	2.477	2.354	2.433	2.347	2.370
7	60	2.470	2.430	2.340	2.347	2.296	2.291
8	70	2.469	2.426	2.326	2.321	2.303	2.282
9	80	2.396	2.305	2.279	2.268	2.249	2.267
10	90	2.339	2.276	2.266	2.305	2.288	2.249
11	100	2.132	2.268	2.248	2.199	2.179	2.213
12	110	2.270	2.249	2.228	2.242	2.181	2.125
13	120	2.215	2.209	2.166	2.202	2.175	2.103
14	130	2.201	2.193	2.119	2.107	2.122	2.103
15	140	2.117	2.150	2.070	2.042	2.089	2.060
16	150	2.083	2.053	2.037	2.034	2.009	1.997
17	160	2.068	2.023	1.992	1.957	1.922	1.865

Table No. 2

Tube Diameter = 3.81 cm, Tangential flow rate = 4040 LPM  
 Air flow rate = 155 LPM, Correction factor for restriction=0.8

Sr. No.	Distance along the length of tube (cm)	Corrected air core diameter (in cm) for superimposed axial flow in (LPM)					
		0	100	200	300	400	500
1	0	2.636	2.375	2.375	2.374	2.259	2.163
2	10	2.653	2.502	2.428	2.349	2.336	2.293
3	20	2.502	2.457	2.367	2.347	2.333	2.257
4	30	2.499	2.432	2.363	2.299	2.293	2.220
5	40	2.434	2.416	2.356	2.316	2.261	2.210
6	50	2.410	2.395	2.393	2.311	2.246	2.215
7	60	2.332	2.363	2.329	2.286	2.230	2.204
8	70	2.363	2.369	2.279	2.262	2.212	2.185
9	80	2.282	2.316	2.230	2.204	2.197	2.175
10	90	2.257	2.267	2.227	2.203	2.170	2.150
11	100	2.163	2.186	2.192	2.093	2.033	2.071
12	110	2.148	2.131	2.175	2.094	2.036	2.054
13	120	2.088	2.032	2.056	2.025	2.006	2.015
14	130	2.061	2.035	2.021	2.023	2.004	1.998

Table No. 8

Tubo diameter = 4.01 cm Tangential flow rate = 4040 LPH

Air flow rate = 600 LPH, Correction factor for retraction = 0.07

Sr. No.	Distance along the length of tube (cm)	Corrected air core diameter (in cm) for superimposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	2.498	2.432	2.460	2.305	2.403	2.334
2	10	2.532	2.571	2.422	2.426	2.366	2.422
3	20	2.528	2.562	2.422	2.415	2.333	2.352
4	30	2.524	2.508	2.424	2.388	2.308	2.320
5	40	2.488	2.455	2.409	2.363	2.326	2.277
6	50	2.453	2.496	2.337	2.349	2.322	2.277
7	60	2.432	2.458	2.362	2.349	2.319	2.240
8	70	2.399	2.245	2.325	2.346	2.312	2.237
9	80	2.366	2.366	2.273	2.339	2.303	2.219
10	90	2.340	2.344	2.262	2.322	2.232	2.203
11	100	2.325	2.307	2.250	2.307	2.262	2.175
12	110	2.325	2.253	2.186	2.302	2.229	2.151
13	120	2.305	2.223	2.164	2.232	2.203	2.129
14	130	2.262	2.227	2.163	2.224	2.212	2.095
15	140	2.245	2.224	2.159	2.175	2.112	2.035

Table No. 9

Tube diameter = 3.81 cm. , Tangential flow rate = 4040 LPH

Air flow rate = 950 LPH , Correction factor for friction = 0.87

Sr No.	Distance along the length of tube (cm)	Corrected air core diameter (in cm) for imposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	2.550	2.526	2.499	2.497	2.575	2.572
2	10	2.579	2.599	2.493	2.499	2.450	2.550
3	20	2.610	2.587	2.520	2.405	2.500	2.270
4	30	2.585	2.553	2.549	2.447	2.99	2.349
5	40	2.589	2.496	2.907	2.968	2.916	2.919
6	50	2.501	2.556	2.599	2.932	2.909	2.224
7	60	2.449	2.446	2.426	2.448	2.900	2.276
8	70	2.442	2.499	2.599	2.536	2.920	2.194
9	80	2.455	2.927	2.905	2.906	2.256	2.192
10	90	2.998	2.931	2.914	2.297	2.220	2.157
11	100	2.901	2.224	2.194	2.173	2.152	2.005
12	110	2.299	2.188	2.209	2.166	2.104	1.984
13	120	2.291	2.248	2.125	2.155	2.152	2.690
14	130	2.276	2.170	2.124	2.165	2.154	1.978
15	140	2.223	2.219	2.169	2.166	2.142	2.029

Table No. 10

Tube Diameter = 3.81 cms. Tangential flow rate = 3344 LPH

Air flow rate = 400 LPH . Correction factor for refraction = 0.87

Sr. No.	Distance along the length of tube (cms)	Corrected air core diameter (in cms) for superimposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	2.460	2.362	2.310	2.311	2.278	2.149
2	10	2.445	2.404	2.459	2.328	2.242	2.219
3	20	2.573	2.339	2.405	2.418	2.292	2.209
4	30	2.900	2.400	2.451	2.306	2.235	2.152
5	40	2.419	2.401	2.341	2.252	2.193	2.175
6	50	2.360	2.324	2.273	2.271	2.205	2.149
7	60	2.401	2.302	2.296	2.246	2.191	2.166
8	70	2.394	2.293	2.241	2.204	2.223	2.145
9	80	2.309	2.243	2.233	2.227	2.185	2.169
10	90	2.172	2.177	2.240	2.217	2.136	2.105
11	100	2.159	2.175	2.118	2.106	2.106	2.1051



Table No. 11

Tube Diameter = 3.81 cms. Tangential flow rate = 3346 LPH

Air flow rate = 500 LPH .Correction factor for refraction=0.67

Sr. No.	Distance along the length of tube (cms)	Corrected air core diameter (in cms) for superimposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	2.422	2.431	2.395	2.135	2.368	2.187
2	10	2.551	2.492	2.487	2.318	2.339	2.329
3	20	2.549	2.502	2.426	2.456	2.311	2.265
4	30	2.543	2.457	2.405	2.317	2.273	2.265
5	40	2.509	2.459	2.393	2.310	2.273	2.225
6	50	2.468	2.419	2.339	2.331	2.294	2.219
7	60	2.443	2.426	2.312	2.256	2.239	2.224
8	70	2.399	2.407	2.311	2.293	2.271	2.214
9	80	2.365	2.353	2.303	2.255	2.261	2.145
10	90	2.313	2.347	2.237	2.239	2.212	2.1001
11	100	2.222	2.184	2.135	2.166	2.123	2.035

Table No. 12

Tube diameter = 3.81 cm, Tangential flow rate = 3944 LPH

Air flow rate = 950 LPH, Correction factor for restriction = 0.87

Sr. No.	Distance along the length of tube (cm)	Corrected air core diameter (in cm) for imposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	2.512	2.356	2.332	2.339	2.366	2.293
2	10	2.533	2.456	2.439	2.396	2.349	2.363
3	20	2.523	2.623	2.464	2.469	2.389	2.412
4	30	2.521	2.546	2.454	2.412	2.346	2.292
5	40	2.430	2.472	2.463	2.293	2.276	2.240
6	50	2.480	2.423	2.390	2.357	2.349	2.266
7	60	2.450	2.375	2.355	2.273	2.260	2.259
8	70	2.447	2.404	2.326	2.367	2.333	2.312
9	80	2.388	2.303	2.270	2.270	2.259	2.200
10	90	2.342	2.310	2.212	2.331	2.253	2.263
11	100	2.179	2.229	2.253	2.256	2.232	2.205

Table No. 11

Tube Diameter = 3.81 cm, Tangential flow rate = 2240 LPH

Air flow rate = 310 LPH, Correction factor for restriction = 0.87

Sr. No.	Distance along the length of tube (cm)	Corrected air diameter (in cm) for sugar imposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	2.426	2.342	2.340	2.092	2.907	
2	10	2.524	2.444	2.393	2.109	2.905	
3	20	2.368	2.329	2.284	2.065	2.876	
4	30	2.360	2.339	2.269	2.152	2.850	
5	40	2.296	2.257	2.246	2.024	2.847	
6	50	2.237	2.244	2.225	2.049	2.770	
7	60	2.248	2.249	2.254	2.024	2.770	
8	70	2.306	2.241	2.275	2.970	2.764	
9	80	2.272	2.201	2.093	2.930	2.749	
10	90	2.262	2.225	2.050	2.902	2.744	

Table No. 14

Tube diameter = 3.81 cm, Tangential flow rate = 2240 LPH

Air flow rate = 600 LPH, Correction factor for restriction = 0.87

Sr. No.	Distance along the length of tube (cm)	Corrected air core diameter (in cm) for superimposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	2.393	2.427	2.392	2.330	2.208	2.185
2	10	2.460	2.399	2.346	2.301	2.239	2.204
3	20	2.537	2.393	2.296	2.272	2.262	2.218
4	30	2.500	2.390	2.309	2.292	2.292	2.184
5	40	2.423	2.332	2.303	2.275	2.023	2.153
6	50	2.427	2.346	2.323	2.312	2.249	2.232
7	60	2.376	2.306	2.234	2.252	2.158	2.130
8	70	2.323	2.282	2.262	2.242	2.123	2.097
9	80	2.312	2.230	2.216	2.134	2.080	2.053
10	90	2.273	2.233	2.177	2.142	2.047	2.002

Table No. 15

Tube diameter = 3.81 cm, Tangential flow rate = 2240 LPH

Air flow rate = 960 LPH, Correction factor for correction = 0.87

Sr.	Distance along the length of tube (cm)	Corrected air core diameter (in cm) for cup-shaped axial flow in (LPH)					
		0	100	200	300	400	500
1	0	2.458	2.415	2.352	2.303	2.299	2.201
2	10	2.466	2.421	2.393	2.352	2.284	2.250
3	20	2.506	2.410	2.387	2.353	2.288	2.255
4	30	2.483	2.403	2.347	2.323	2.279	2.245
5	40	2.452	2.393	2.321	2.306	2.262	2.230
6	50	2.426	2.412	2.306	2.262	2.247	2.270
7	60	2.362	2.354	2.353	2.240	2.258	2.163
8	70	2.338	2.307	2.292	2.243	2.215	2.132
9	80	2.321	2.304	2.210	2.207	2.189	2.125
10	90	2.297	2.281	2.219	2.196	2.178	2.133

Table No. 15

Tubo Diameter = 9.715 cms.

Tangential flow rate = 6164 LPH

Air flow rate = 950 LPH

Correction factor for refraction = 0.89

Sr. No.	Distance along the length of tube (cms)	Corrected air core diameter (in cms) for superimposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	5.1860	5.102	5.042	4.924	4.880	4.793
2	10	5.175	5.100	4.932	4.910	4.852	4.738
3	20	5.148	5.050	4.950	4.880	4.821	4.775
4	30	5.070	5.000	4.874	4.834	4.785	4.756
5	40	4.940	4.900	4.750	4.710	4.682	4.635
6	50	4.880	4.802	4.702	4.658	4.584	4.520
7	60	4.774	4.752	4.620	4.563	4.512	4.454
8	70	4.680	4.612	4.580	4.525	4.484	4.444
9	80	4.620	4.602	4.500	4.452	4.410	4.410
10	90	4.557	4.510	4.480	4.426	4.392	4.376
11	100	4.500	4.450	4.452	4.401	4.372	4.356
12	110	4.510	4.448	4.402	4.389	4.366	4.342
13	120	4.454	4.412	4.419	4.350	4.342	4.333
14	130	4.450	4.390	4.360	4.325	4.316	4.310
15	140	4.442	4.390	4.342	4.318	4.299	4.392
16	150	4.362	4.362	4.340	4.302	4.282	4.233

Table No. 17

Tube diameter = 5.719  
 Tangential flow rate = 6254 LPH  
 Air flow rate = 710 LPH  
 Correction factor for correction = 0.85

Sr. No.	Distance along the length of tube (cm)	Corrected air core diameter (in cm) for superimposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	5.210	5.192	5.120	5.080	4.930	4.815
2	10	5.275	5.162	5.102	5.054	4.935	4.810
3	20	5.124	5.112	5.062	5.028	4.890	4.800
4	30	5.080	5.030	4.910	4.925	4.884	4.788
5	40	5.022	4.932	4.946	4.910	4.832	4.776
6	50	4.910	4.900	4.862	4.821	4.780	4.720
7	60	4.800	4.762	4.722	4.684	4.620	4.572
8	70	4.734	4.700	4.672	4.622	4.580	4.514
9	80	4.625	4.610	4.575	4.525	4.510	4.480
10	90	4.583	4.552	4.520	4.476	4.465	4.420
11	100	4.564	4.552	4.520	4.461	4.442	4.411
12	110	4.540	4.520	4.460	4.410	4.400	4.392
13	120	4.500	4.488	4.432	4.412	4.374	4.385
14	130	4.482	4.456	4.412	4.375	4.346	4.376
15	140	4.440	4.426	4.400	4.363	4.329	4.350
16	150	4.420	4.405	4.390	4.362	4.348	4.332

Table No. 18

Tube diameter = 5.715 cm  
 Tangential flow rate = 6164 LPH  
 Air flow rate = 1200 LPH  
 Correction factor for reflection = 0.85

F. No.	Distance along the length of tube (cm)	Corrected air core diameter (in cm) for superimposed axial flow in (LPH)					
		0	100	200	3000	4000	500
1	0	5.256	5.238	5.280	5.122	4.968	4.880
2	10	5.210	5.162	5.110	5.074	4.922	4.835
3	20	5.181	5.125	5.078	5.018	4.874	4.820
4	30	5.126	5.082	4.962	4.914	4.823	4.813
5	40	5.082	5.024	4.912	4.875	4.786	4.762
6	50	4.924	4.902	4.854	4.812	4.740	4.700
7	60	4.873	4.832	4.728	4.683	4.675	4.590
8	70	4.812	4.765	4.714	4.679	4.612	4.580
9	80	4.714	4.682	4.634	4.592	4.544	4.500
0	90	4.610	4.572	4.600	4.534	4.510	4.447
1	100	4.590	4.529	4.577	4.522	4.475	4.424
2	110	4.582	4.522	4.519	4.480	4.424	4.400
3	120	4.540	4.490	4.512	4.460	4.428	4.391
4	130	4.525	4.472	4.436	4.414	4.400	4.385
5	140	4.514	4.469	4.462	4.432	4.399	4.381
6	150	4.492	4.463	4.440	4.420	4.393	4.362



Table No. 19

Tube diameter = 5.715 cm  
 Tangential flow rate = 9555 LPH  
 Air flow rate = 950 LPH  
 Correction factor for refraction = 0.05

Sr. No.	Distance along the length of tube (cm)	Corrected air core diameter (in cm) for superimposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	5.110	5.034	4.956	4.882	4.801	4.727
2	10	5.050	4.930	4.922	4.848	4.766	4.692
3	20	5.016	4.938	4.878	4.810	4.728	4.659
4	30	4.938	4.882	4.839	4.768	4.694	4.620
5	40	4.876	4.835	4.734	4.692	4.658	4.585
6	50	4.821	4.800	4.684	4.593	4.530	4.502
7	60	4.770	4.706	4.606	4.544	4.503	4.438
8	70	4.666	4.601	4.532	4.520	4.474	4.410
9	80	4.612	4.575	4.490	4.443	4.406	4.373
10	90	4.542	4.490	4.430	4.422	4.392	4.350
11	100	4.494	4.412	4.423	4.400	4.332	4.339
12	110	4.437	4.419	4.400	4.336	4.341	4.294
13	120	4.411	4.332	4.401	4.350	4.309	4.250
14	130	4.378	4.334	4.330	4.294	4.233	4.202
15	140	4.323	4.355	4.309	4.238	4.216	4.164
16	150	4.200	4.314	4.233	4.214	4.172	4.123

Table NO. 20

Tube diameter 5.715 cm  
 Tangential flow rate = 5555 LPH  
 Air flow rate = 710 LPH  
 Correction factor for refraction = 0.85

Sr. No.	Distance along the length of tube (cms)	Corrected air core diameter (in cms) for superimposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	5.125	5.062	5.000	4.926	4.850	4.749
2	10	5.080	4.975	4.924	4.859	4.800	4.712
3	20	4.992	4.926	4.868	4.817	4.764	4.678
4	30	4.885	4.838	4.810	4.770	4.723	4.647
5	40	4.801	4.839	4.774	4.726	4.684	4.605
6	50	4.730	4.793	4.729	4.683	4.645	4.569
7	60	4.684	4.743	4.684	4.638	4.604	4.533
8	70	4.654	4.681	4.642	4.594	4.554	4.493
9	80	4.621	4.601	4.576	4.509	4.522	4.460
10	90	4.574	4.550	4.524	4.434	4.438	4.402
11	100	4.559	4.5417	4.508	4.442	4.440	4.383
12	110	4.526	4.510	4.460	4.409	4.393	4.353
13	120	4.494	4.486	4.422	4.370	4.351	4.317
14	130	4.453	4.441	4.378	4.330	4.310	4.283
15	140	4.422	4.393	4.332	4.286	4.275	4.245
16	160	4.376	4.354	4.290	4.243	4.210	4.158

Table No. 21

Tube Diameter = 5.715 cm  
 Tangential flow rate = 5555 LPH  
 Air flow rate = 1200 LPH  
 Correction factor for restriction = 0.85

Sr. No.	Distance along the length of tube (cm)	Corrected air core diameter (in cm) for superimposed axial flow in LPH					
		0	100	200	300	400	500
1	0	5.153	5.102	5.049	4.978	4.895	4.776
2	10	4.938	5.065	5.000	4.933	4.864	4.720
3	20	4.912	4.992	4.952	4.888	4.818	4.663
4	30	4.884	4.464	4.904	4.834	4.770	4.652
5	40	4.841	4.910	4.853	4.791	4.727	4.614
6	50	4.802	4.834	4.808	4.742	4.683	4.574
7	60	4.769	4.810	4.730	4.683	4.600	4.534
8	70	4.723	4.770	4.712	4.651	4.592	4.497
9	80	4.684	4.723	4.664	4.604	4.565	4.457
10	90	4.643	4.674	4.615	4.550	4.512	4.410
11	100	4.607	4.630	4.573	4.510	4.467	4.334
12	110	4.564	4.638	4.524	4.469	4.426	4.331
13	120	4.523	4.523	4.472	4.416	4.333	4.302
14	130	4.480	4.498	4.420	4.338	4.241	4.231
15	140	4.452	4.442	4.376	4.320	4.293	4.222
16	150	4.412	4.316	4.323	4.276	4.254	4.188

Table 23

Tube Diameter	= 5.725 cm
Tangential flow rate	= 4945 LPH
Air flow rate	= 950 LPH
Correction Factor for restriction	= 0.85

Sr. No.	Distance along the length of tube (cm)	Corrected air core diameter (in cm) for unimpeded axial flow in (LPH)					
		0	100	200	300	400	500
1	0	4.994	4.950	4.912	4.860	4.775	4.639
2	10	4.943	4.896	4.860	4.788	4.722	4.641
3	20	4.897	4.847	4.810	4.738	4.674	4.591
4	30	4.848	4.800	4.759	4.689	4.629	4.544
5	40	4.802	4.750	4.707	4.635	4.560	4.473
6	50	4.752	4.702	4.662	4.584	4.513	4.426
7	60	4.704	4.657	4.612	4.537	4.463	4.373
8	70	4.659	4.603	4.560	4.483	4.411	4.310
9	80	4.603	4.555	4.509	4.432	4.358	4.256
10	90	4.563	4.505	4.461	4.380	4.307	4.205
11	100	4.512	4.459	4.413	4.332	4.254	4.152
12	110	4.464	4.403	4.361	4.281	4.202	4.100
13	120	4.417	4.356	4.312	4.229	4.150	4.048
14	130	4.365	4.313	4.265	4.175	4.099	3.995
15	140	4.320	4.260	4.211	4.126	4.042	3.938
16	150	4.272	4.212	4.160	4.075	3.990	3.882

Table No. 23

Tube diameter = 9.715 cm  
 Tangential flow rate = 4945 LPH  
 Air flow rate = 710 LPH  
 Correction factor for restriction = 0.05

Sr. No.	Distance along the length of tube (cm)	Corrected air core diameter (in cm) for superimposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	5.040	4.978	4.936	4.894	4.730	4.692
2	10	4.992	4.931	4.888	4.808	4.730	4.637
3	20	4.944	4.886	4.840	4.757	4.684	4.597
4	30	4.897	4.839	4.799	4.703	4.633	4.552
5	40	4.849	4.790	4.744	4.656	4.584	4.503
6	50	4.802	4.744	4.693	4.612	4.535	4.459
7	60	4.753	4.693	4.644	4.561	4.487	4.412
8	70	4.702	4.646	4.600	4.514	4.437	4.362
9	80	4.656	4.602	4.552	4.463	4.389	4.316
10	90	4.602	4.555	4.500	4.416	4.340	4.263
11	100	4.560	4.509	4.456	4.367	4.293	4.220
12	110	4.512	4.460	4.402	4.312	4.240	4.173
13	120	4.465	4.413	4.360	4.265	4.193	4.127
14	130	4.419	4.367	4.317	4.219	4.147	4.080
15	140	4.363	4.320	4.264	4.163	4.093	4.037
16	150	4.320	4.269	4.210	4.120	4.040	3.982

Table No. 24

Tube diameter = 5.715 cm  
 Tangential flow rate = 4949 LPH  
 Air flow rate = 1200 LPH  
 Correction factor for porosity = 0.85

Sr. No.	Distance along the length of tube (cm)	Corrected air core diameter (in cm) for unimpeded axial flow in (LPH)					
		0	100	200	300	400	500
1	0	5.090	5.030	4.963	4.880	4.795	4.702
2	10	5.046	4.926	4.922	4.829	4.743	4.659
3	20	4.992	4.932	4.875	4.787	4.700	4.620
4	30	4.948	4.889	4.823	4.739	4.652	4.565
5	40	4.900	4.842	4.780	4.692	4.603	4.522
6	50	4.852	4.792	4.739	4.642	4.559	4.470
7	60	4.806	4.742	4.679	4.596	4.520	4.423
8	70	4.753	4.700	4.647	4.554	4.469	4.376
9	80	4.703	4.652	4.603	4.500	4.419	4.329
10	90	4.640	4.606	4.553	4.460	4.370	4.283
11	100	4.617	4.560	4.513	4.414	4.322	4.232
12	110	4.563	4.511	4.460	4.365	4.276	4.186
13	120	4.520	4.462	4.423	4.322	4.230	4.140
14	130	4.462	4.409	4.373	4.272	4.185	4.092
15	140	4.420	4.376	4.336	4.230	4.136	4.047
16	150	4.372	4.322	4.284	4.184	4.089	4.000

Table No. 25

Tube diameter	= 5.715 cm
Tangential flow rate	= 4500 LPH
Air flow rate	= 990 LPH
Correction factor for refraction	= 0.85

Sr. No.	Distance along the length of tube (cm)	Corrected air core diameter (in cm) for compensated axial flow in (LPH)					
		0	100	200	300	400	500
1	0	4.950	4.900	4.861	4.782	4.703	4.660
2	10	4.900	4.851	4.797	4.729	4.657	4.621
3	20	4.843	4.793	4.741	4.678	4.590	4.563
4	30	4.783	4.742	4.690	4.622	4.545	4.510
5	40	4.742	4.689	4.633	4.570	4.493	4.453
6	50	4.691	4.634	4.580	4.519	4.437	4.393
7	60	4.642	4.583	4.527	4.464	4.383	4.343
8	70	4.589	4.523	4.470	4.411	4.329	4.289
9	80	4.533	4.472	4.420	4.357	4.273	4.230
10	90	4.480	4.417	4.366	4.308	4.223	4.169
11	100	4.431	4.367	4.310	4.249	4.165	4.110
12	110	4.380	4.310	4.258	4.195	4.112	4.053
13	120	4.323	4.253	4.204	4.141	4.059	4.000

Table No. 26

Tube diameter = 5.715 cms  
 Tangential flow rate = 4500 LPH  
 Air flow rate = 710 LPH  
 Correction factor for refraction = 0.85

Sr. No.	Distance along the length of tube (cm)	Corrected air core diameter (in cms) for superimposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	4.950	4.900	4.851	4.782	4.703	4.680
2	10	4.900	4.851	4.797	4.729	4.657	4.624
3	20	4.843	4.793	4.741	4.678	4.590	4.568
4	30	4.786	4.742	4.680	4.622	4.545	4.510
5	40	4.742	4.689	4.633	4.570	4.493	4.456
6	50	4.691	4.623	4.580	4.519	4.437	4.400
7	60	4.642	4.583	4.527	4.464	4.383	4.343
8	70	4.589	4.526	4.478	4.411	4.329	4.239
9	80	4.536	4.472	4.420	4.357	4.273	4.230
10	90	4.480	4.417	4.366	4.308	4.226	4.169
11	100	4.431	4.367	4.310	4.249	4.165	4.110
12	110	4.380	4.310	4.258	4.195	4.112	4.056
13	120	4.328	4.256	4.204	4.141	4.059	4.000



Table No. 22

Tube diameter	= 6.715 cms
Tangential flow rate	= 4500 LPH
Air flow rate	= 1200 LPH
Correction factor for refraction	= 0.85

Sr. No.	Distance along the length of tube (cms)	Corrected air core diameter (in cms) for superimposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	5.032	4.958	4.900	4.823	4.711	4.638
2	10	4.983	4.919	4.854	4.775	4.665	4.592
3	20	4.935	4.872	4.810	4.729	4.621	4.549
4	30	4.884	4.821	4.762	4.679	4.573	4.500
5	40	4.832	4.776	4.716	4.631	4.527	4.453
6	50	4.780	4.723	4.674	4.585	4.480	4.412
7	60	4.733	4.682	4.628	4.531	4.435	4.366
8	70	4.682	4.620	4.579	4.482	4.386	4.312
9	80	4.623	4.571	4.532	4.433	4.339	4.264
10	90	4.570	4.524	4.488	4.387	4.293	4.217
11	100	4.522	4.471	4.436	4.332	4.243	4.178
12	110	4.473	4.421	4.391	4.289	4.199	4.123
13	120	4.425	4.371	4.346	4.240	4.152	4.076

Table No. 28

Tube diameter = 5.735 cm  
 Tangential flow rate = 4500 LPH  
 Air flow rate = 950 LPH  
 Correction factor for restriction = 0.85

Sr. No.	Distance along the length of tube (cm)	Corrected air core diameter (in cm) for superimposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	4.926	4.900	4.782	4.710	4.652	
2	10	4.866	4.839	4.723	4.652	4.598	
3	20	4.809	4.775	4.666	4.590	4.530	
4	30	4.752	4.700	4.607	4.538	4.473	
5	40	4.698	4.632	4.549	4.478	4.412	
6	50	4.623	4.554	4.479	4.420	4.358	
7	60	4.566	4.512	4.438	4.368	4.298	
8	70	4.506	4.450	4.370	4.310	4.232	
9	80	4.442	4.390	4.312	4.245	4.180	
10	90	4.380	4.325	4.253	4.180	4.121	

DISTURBANCES

Table No. 29

Tube Diameter	= 5.715 mm
Tangential flow rate	= 3500 LPH
Air flow rate	= 710 LPH
Correction factor for refraction	= 0.85

Sr. No.	Distance along the length of tube (mm)	Corrected air core diameter (in mm) for superimposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	4.950	4.934	4.812	4.747	4.632	4.630
2	10	4.899	4.872	4.755	4.699	4.625	4.579
3	20	4.839	4.812	4.692	4.637	4.562	4.517
4	30	4.775	4.745	4.630	4.579	4.513	4.452
5	40	4.712	4.680	4.569	4.512	4.452	4.393
6	50	4.651	4.615	4.522	4.458	4.397	4.345
7	60	4.599	4.556	4.470	4.400	4.348	4.293
8	70	4.532	4.490	4.412	4.340	4.280	4.220
9	80	4.481	4.432	4.352	4.284	4.229	4.166
10	90	4.422	4.365	4.293	4.226	4.163	4.100

Table No. 30

Tube diameter	= 5.715 cm
Tangential flow rate	= 3500 LPH
Air flow rate	= 1200 LPH
Correction factor for refraction	= 0.85

Sr. No.	Distance along the length of tube (cm)	Corrected air core diameter (in cm) for superimposed axial flow in (LPH)					
		0	100	200	300	400	500
1	0	4.998	4.926	4.762	4.712	4.663	4.581
2	10	4.440	4.872	4.721	4.671	4.613	4.534
3	20	4.872	4.819	4.638	4.626	4.569	4.487
4	30	4.829	4.755	4.632	4.595	4.520	4.445
5	40	4.769	4.690	4.593	4.538	4.465	4.393
6	50	4.708	4.631	4.557	4.490	4.410	4.343
7	60	4.650	4.584	4.512	4.446	4.363	4.296
8	70	4.599	4.522	4.473	4.403	4.315	4.254
9	80	4.532	4.470	4.430	4.356	4.264	4.265
10	90	4.472	4.413	4.392	4.311	4.210	4.153

**CHAPTER IV**

**DISCUSSION OF RESULTS CONCLUSIONS AND  
RECOMMENDATIONS**

TUBE DIAMETER = 3.81 cms  
 TANGENTIAL FLOW RATE = 3344 L.P.H.

SYMBOL	SUPERIMPOSED AXIAL FLOW IN L.P.H.
○	0
x	100
⊗	200
△	300
○	400
□	500

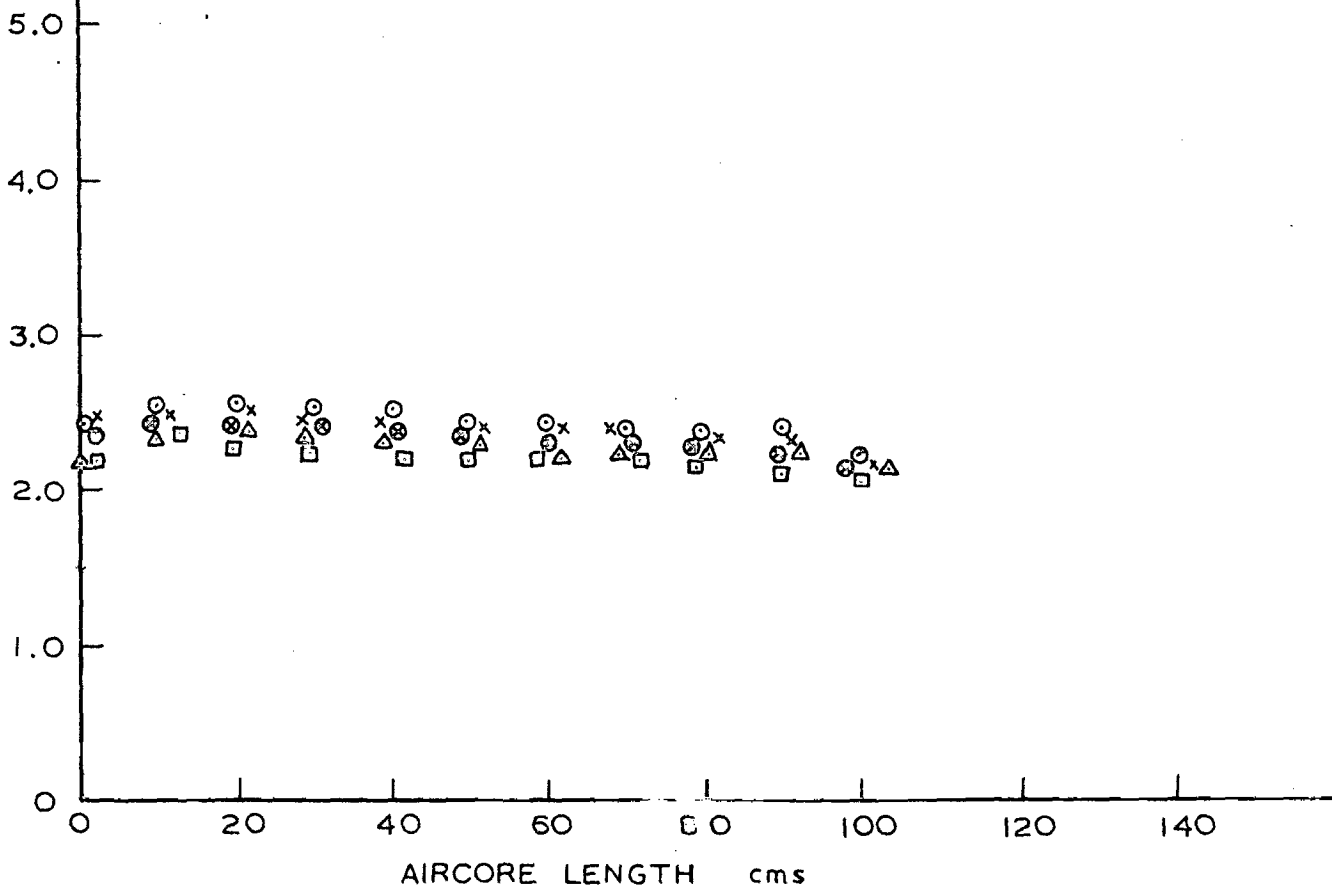
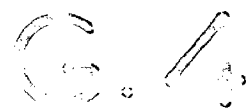
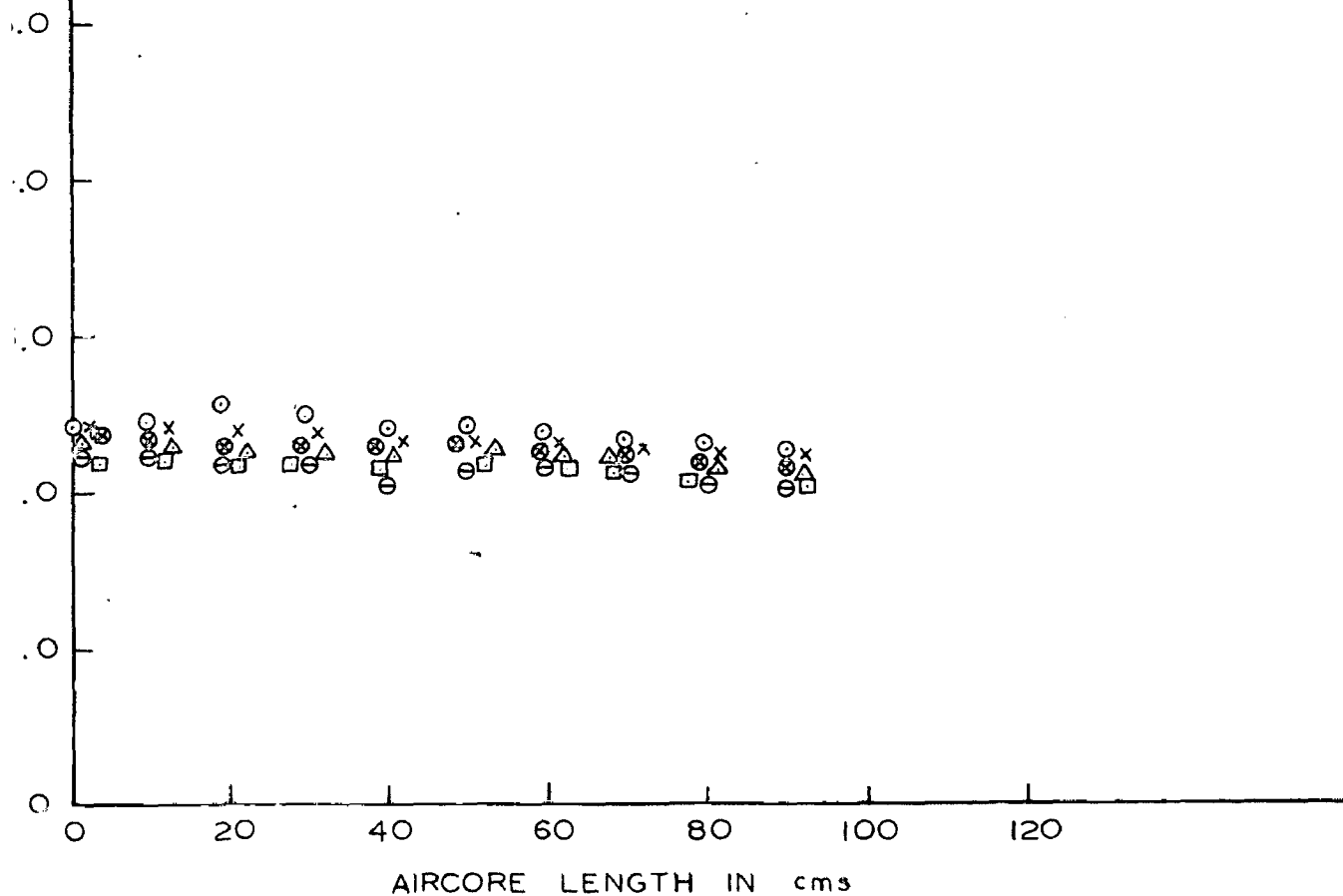


FIG. 3

VARIATION OF AIRCORE DIAMETER ALONG THE  
 AIRCORE LENGTH FOR DIFFERENT SUPERIMPOSED  
 AIR RATES

TUBE DIAMETER = 3.81 cms  
 TANGENTIAL FLOW RATE = 2240 L.P.H.

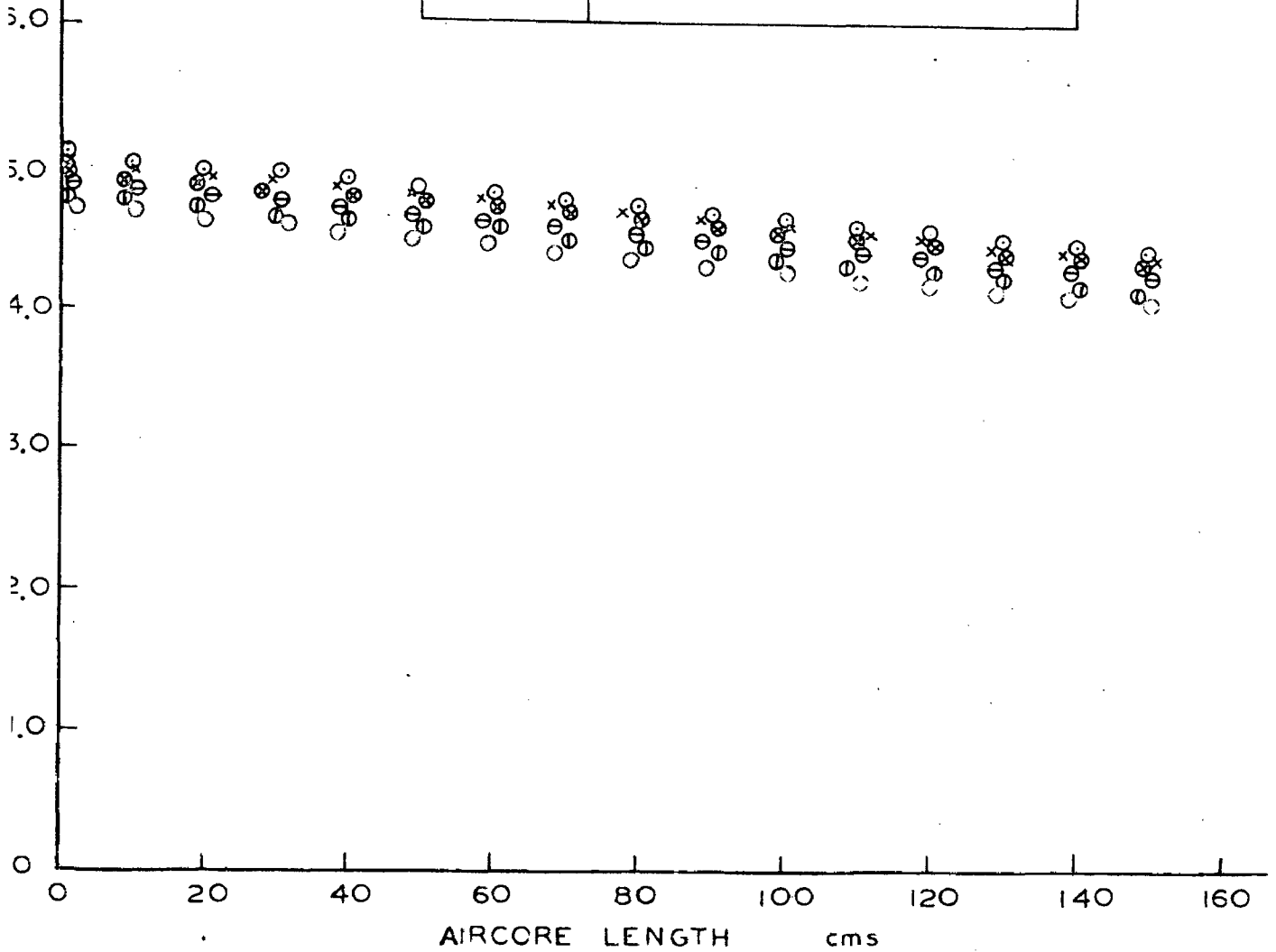
SYMBOL	SUPERIMPOSED AXIAL FLOW IN L.P.H.
○	0
X	100
⊗	200
△	300
⊖	400
□	500



VARIATION OF AIRCORE DIAMETER ALONG THE AIRCORE LENGTH FOR DIFFERENT SUPERIMPOSED AIR RATES

TUBE DIAMETER = 5.715 cms  
 TANGENTIAL FLOW RATE = 4945 L.P.H.  
 AXIAL FLOW RATE = 1200 L.P.H.

SYMBOL	SUPERIMPOSED AXIAL FLOW IN L.P.H.
○	0
X	100
⊗	200
⊖	300
⊕	400
○	500

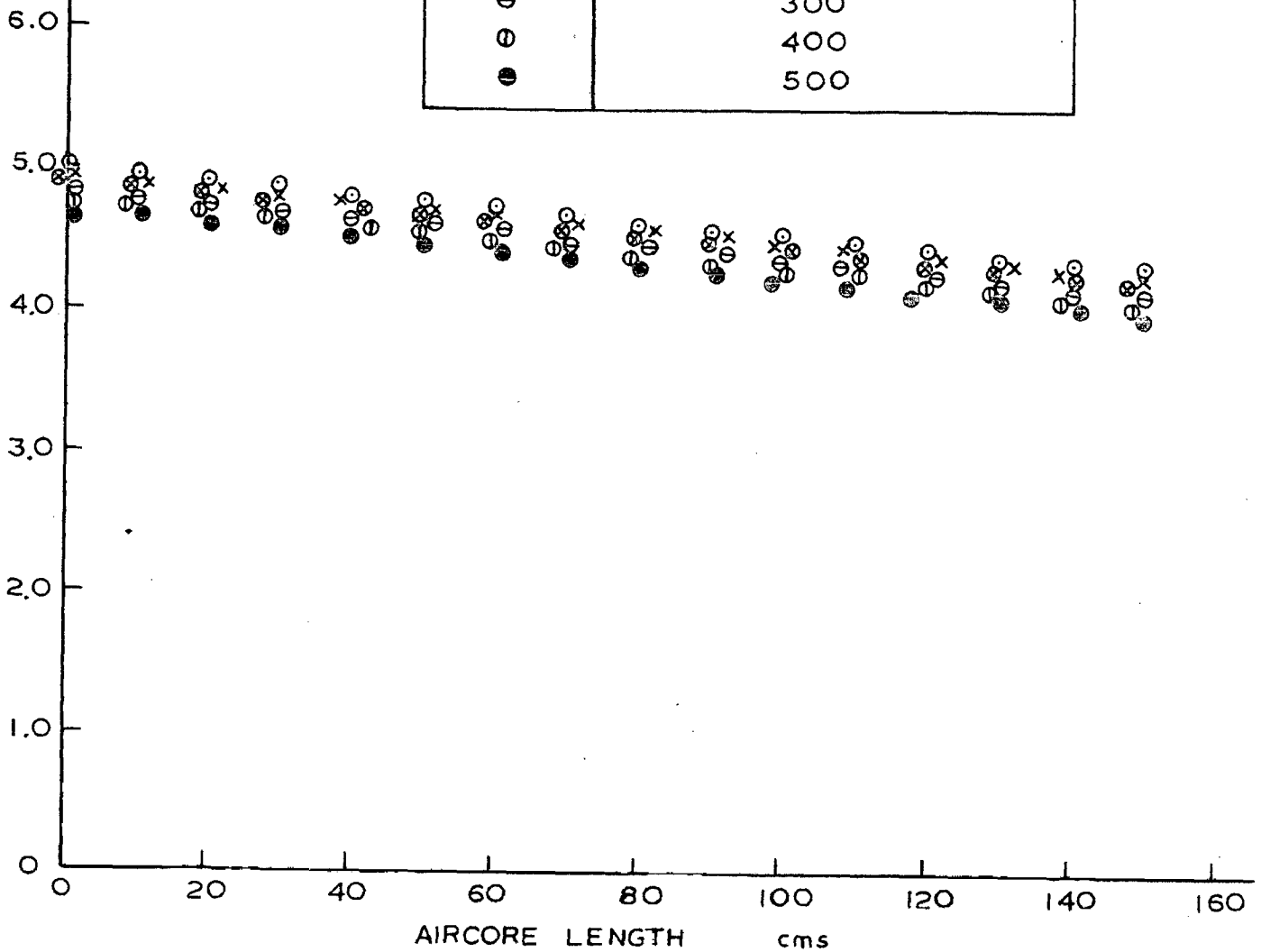


VARIATION OF AIRCORE DIAMETER ALONG THE AIRCORE LENGTH FOR DIFFERENT SUPERIMPOSED AXIAL FLOW RATES



TUBE DIAMETER = 5.715 cms  
 TANGENTIAL FLOW RATE = 4945 L.P.H.  
 AIR FLOW RATE = 330 L.P.H.

SYMBOL	SUPERIMPOSED AXIAL FLOW IN L.P.H.
○	0
X	100
⊗	200
⊖	300
⊕	400
●	500



**FIG. 6**

VARIATION OF AIRCORE DIAMETER ALONG THE  
 AIRCORE LENGTH FOR DIFFERENT SUPERIMPOSED  
 AIR RATES

## CHAPTER XII

### DISCUSSION OF RESULTS AND CONCLUSIONS

The studies on the superimposition of axial flow on swirling flow have been carried out on tubes of diameter 3.81 cm and 5.715 cm. The results obtained from the data stated in chapter III are plotted. The representative plots of the same are attached.

Figure 3 and 4 show the variation of air core diameter along the air-core length at different superimposed axial flows while keeping the tangential flow and air rate constant a value. These are the representative graphs obtained by plotting data for proper tube of diameter 3.81 cm at higher tangential flow rates of water. Other data on this tube diameter also gave similar plots. It can be clearly seen from the graphs that the air-core diameter while superimposing different axial flows does not change appreciably from the air core diameter without any superimposition of axial flow under the experimental conditions and limits used. Similar type of representative plots obtained from the data while using the tube of 5.715 cm diameter are shown in Figure 5 and 6.

Figure 7 shows the variation of average air core diameter with different superimposed axial flows, using different air rates. From the figure it is clear that the

TUBE DIAMETER=3.81CMS  
 TANGENTIAL FLOW RATE=3344 L.P.H.

SYMBOL	AIR FLOW RATE IN L.P.H.
○	400
⊗	580
○	960

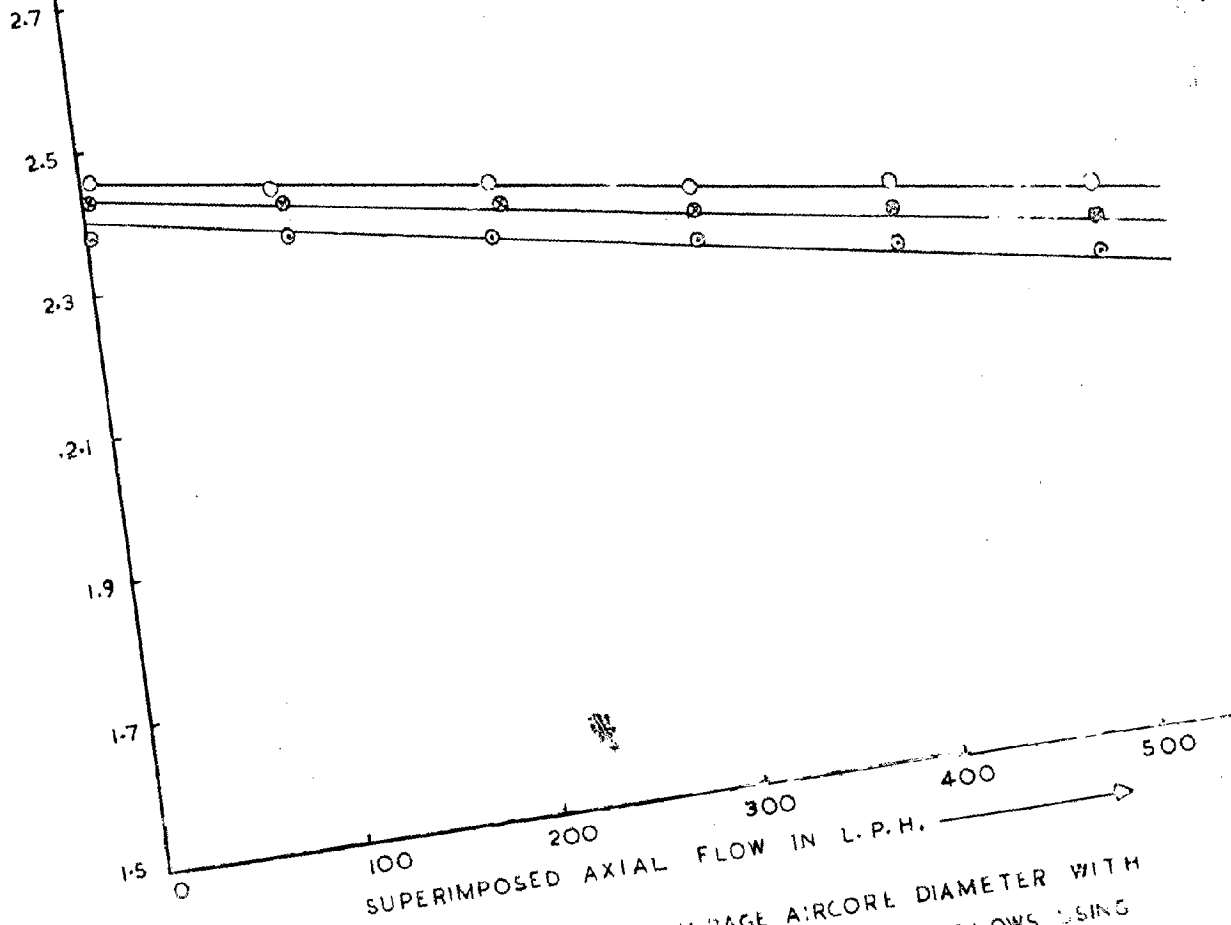


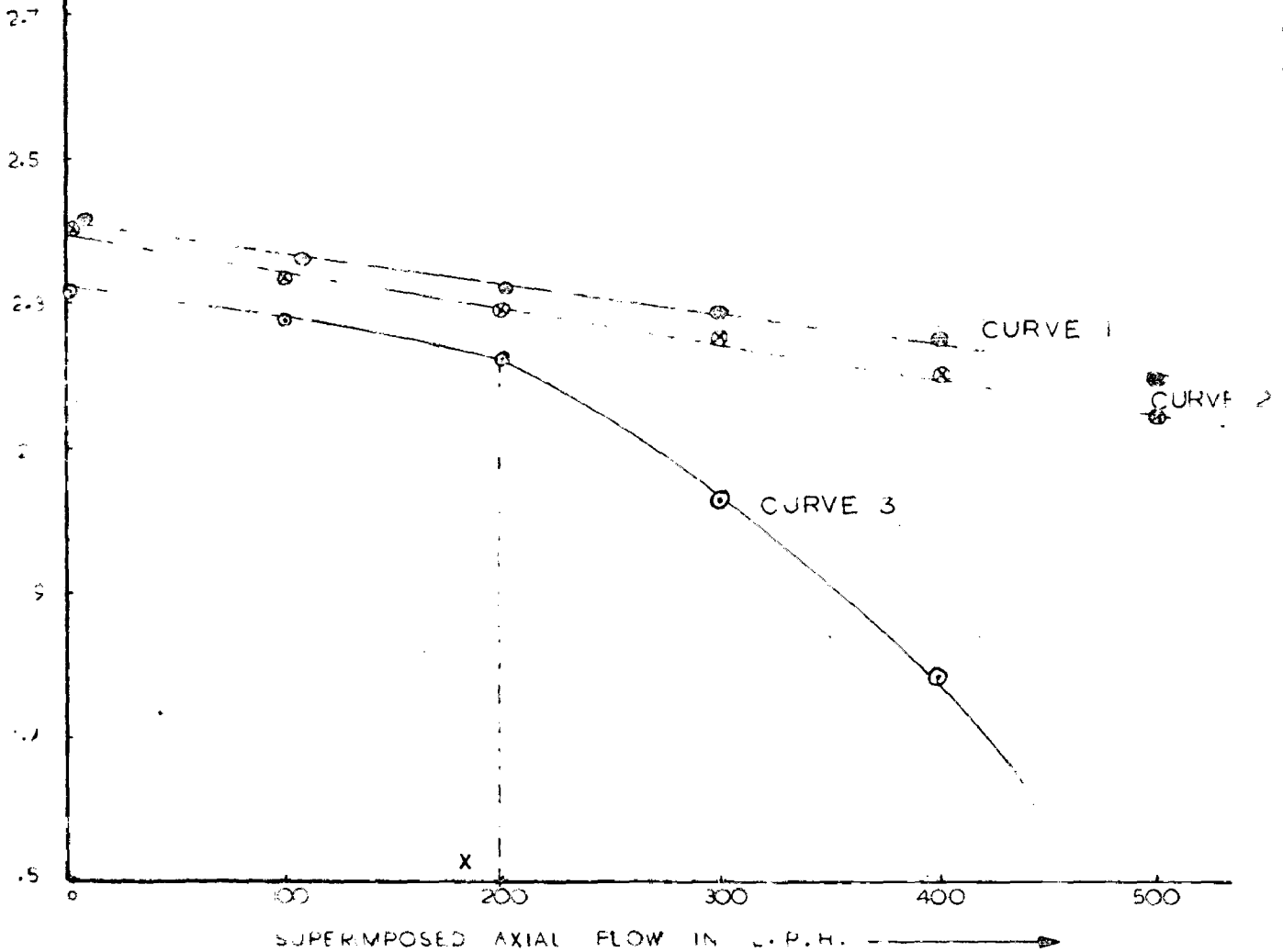
Fig. 7

VARIATION OF AVERAGE AIRCORE DIAMETER WITH  
 DIFFERENT SUPERIMPOSED AXIAL FLOWS USING  
 DIFFERENT AIR RATES

TUBE DIAMETER = 3.81 CM S

TANGENTIAL FLOW RATE = 2240 L.P.H.

SYMBOL	AIR FLOW RATE IN L.P.H.
○	310
⊗	600
⊕	960



**FIG. 8**

VARIATION OF AVERAGE AIR CORE DIAMETER WITH DIFFERENT SUPERIMPOSED AXIAL FLOWS USING DIFFERENT AIR RATES

TUBE DIAMETER = 5.275 CMS

TANGENTIAL FLOW RATE = 4945 L.P.H.

SYMBOL	AIR FLOW RATE IN L.P.H.
○	330
⊗	710
○	1200

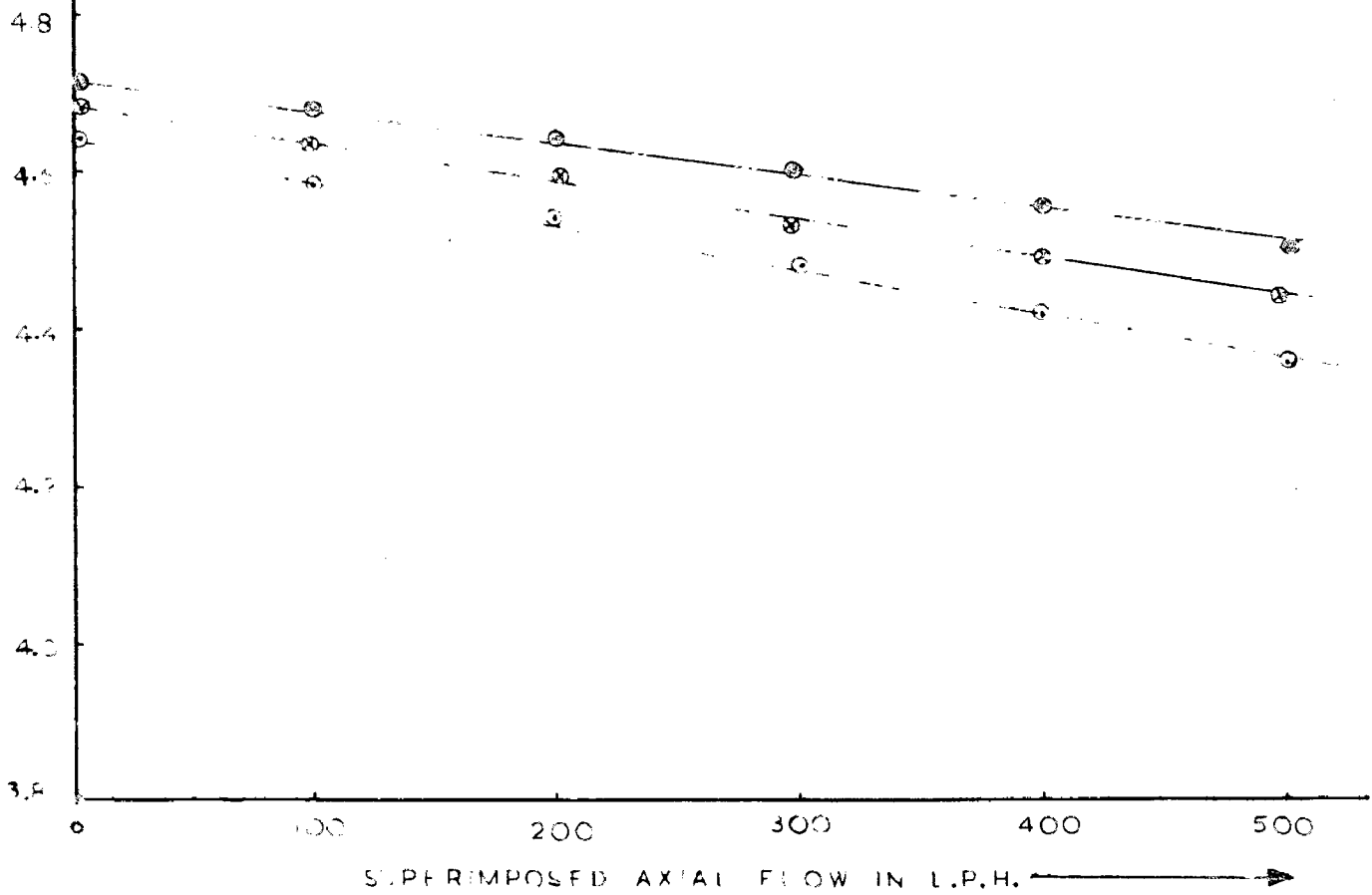


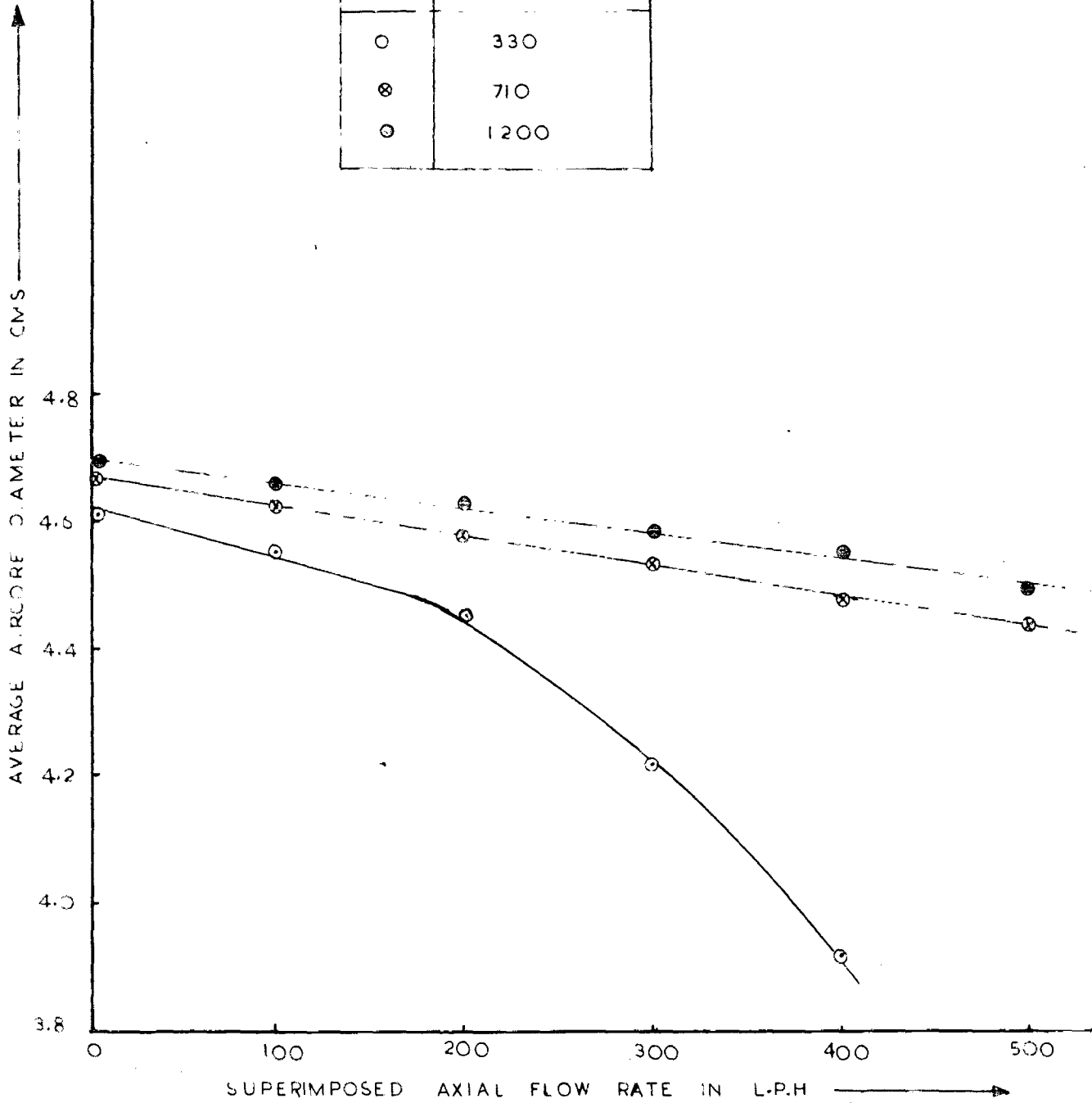
FIG. 9

VARIATION OF AVERAGE AIRCORE DIAMETER WITH DIFFERENT SUPERIMPOSED AXIAL FLOWS USING DIFFERENT AIR RATES

TUBE DIAMETER = 5.75 CMS

TANGENTIAL FLOW RATE = 3600 L.P.H.

SYMBOL	AIR FLOW RATE IN L.P.H.
○	330
⊗	710
⊙	1200



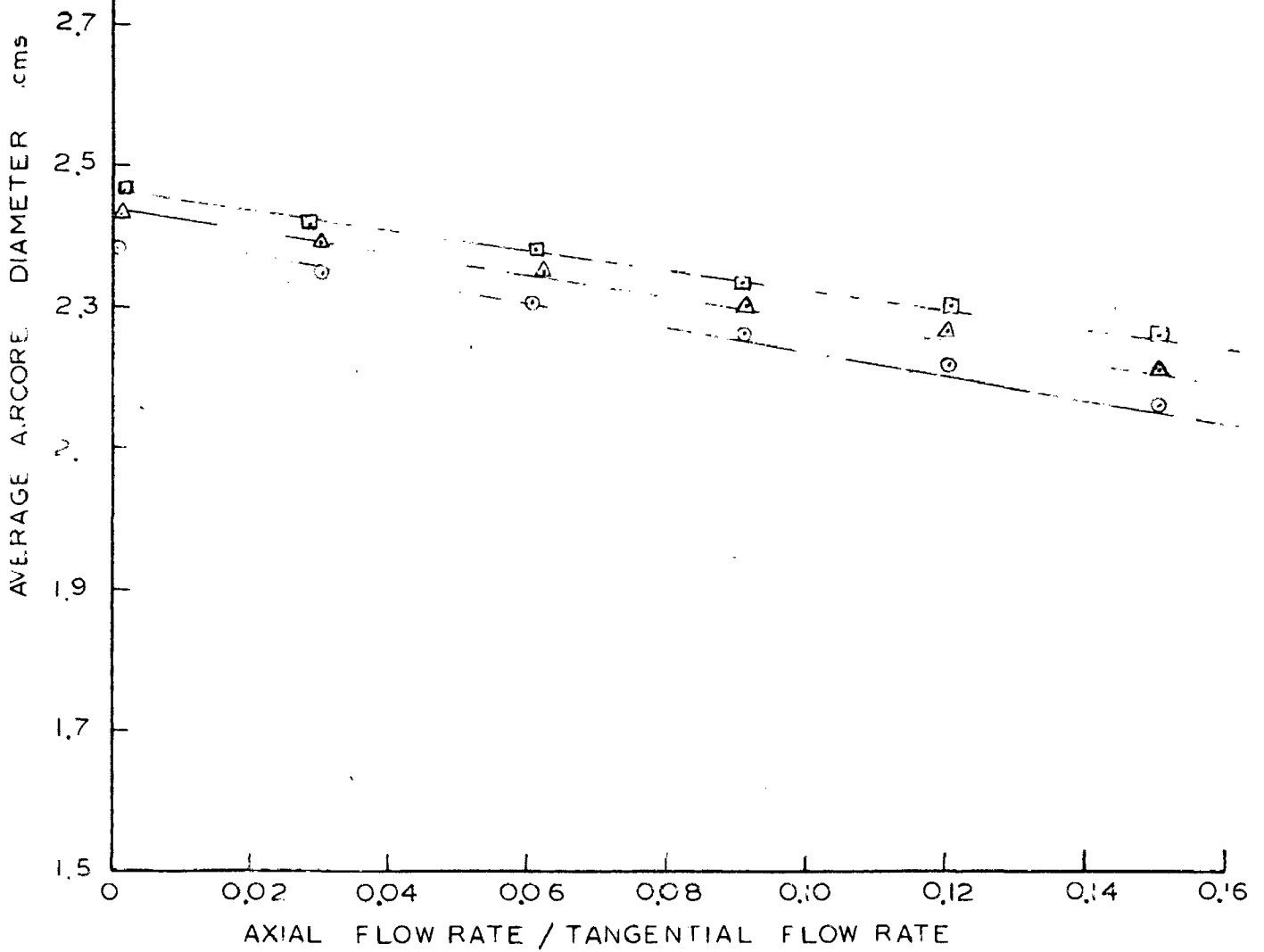
**FIG 10** VARIATION OF AVERAGE AIRCORE DIAMETER WITH DIFFERENT SUPERIMPOSED AXIAL FLOWS USING DIFFERENT AIR RATES

average air-core diameter decreases as the superimposed axial flow increases. It can also be seen from the figure that the slope of curves keeps on increasing with decreasing air rates, which reveals that the rate of decrease in average air-core diameter with superimposed axial flow rates is more at low air rates. Figure 8 also shows the variation of average air core diameter for superimposed axial flow rates using different air rates. The tangential flow rate in case of Fig. 7 is more than that of in case of Fig. 8. From Figure 8 we find that at low tangential flow rates the rate of decrease of air core diameter is much more rapid as compared to those at higher tangential flow rates in case of low air rates. This implies that as the tangential flow rates increases the rate of decrease of average air core diameter for different superimposed axial flows decreases.

From Fig. 8 curve 3 we observe that point  $x$  such that the slope of the curve corresponding to axial flows more than  $x$  is greater than the slope corresponding to axial flows less than  $x$ . The value of axial flow rate at  $x$  for this particular tangential flow rate, tube diameter and air rate can be called as critical axial flow, which is the maximum axial flow which can be superimposed at this value of tangential flow without disturbing the air core. If we take average air core diameter corresponding to 500 RPM of axial flow

TUBE DIAMETER = 3.81 cms  
 TANGENTIAL FLOW RATE = 3344 L.P.H.

SYMBOL	AIR FLOW RATE L.P.H.
○	400
△	580
□	960



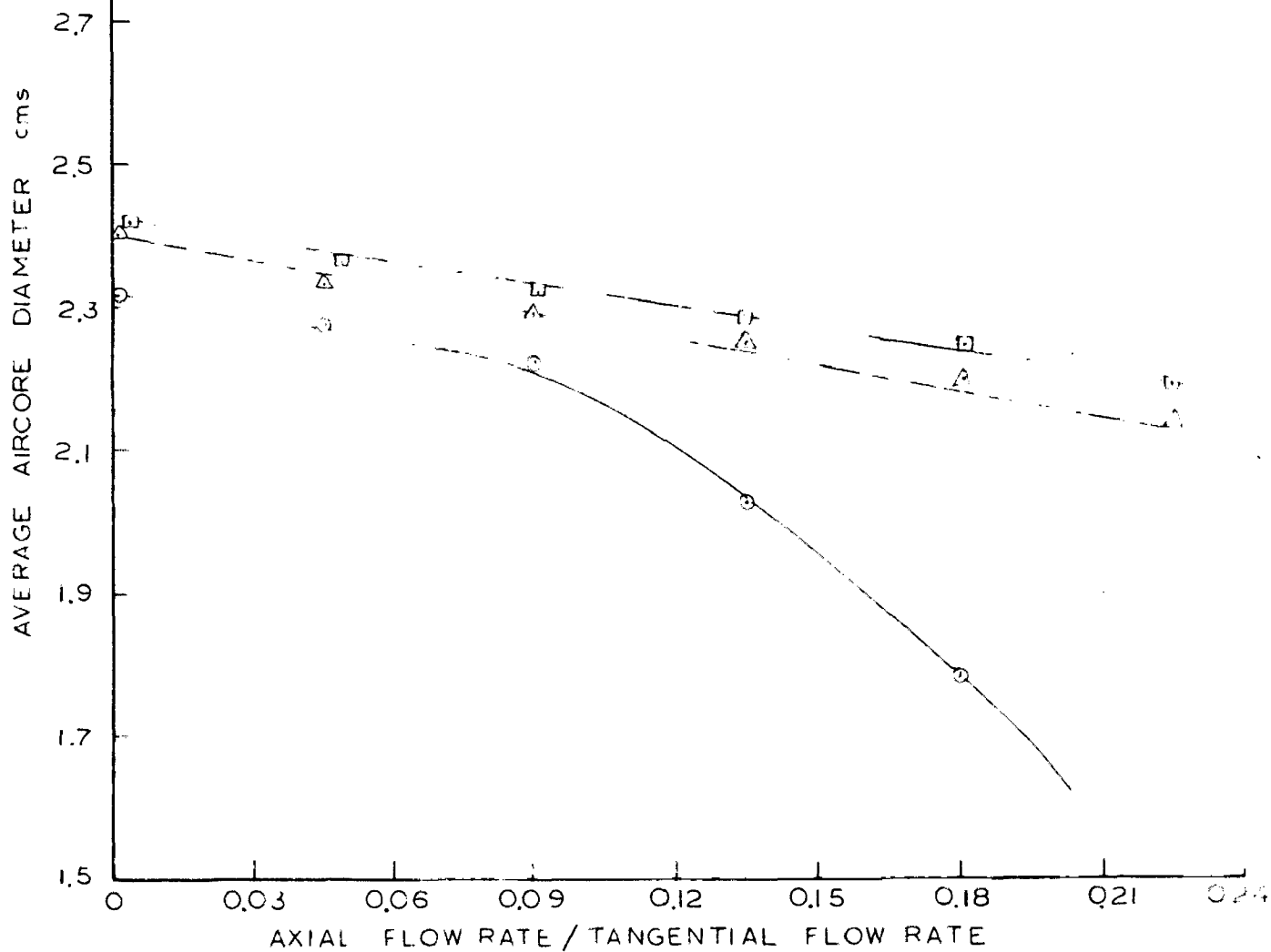
**FIG. 1**

VARIATION OF AVERAGE AIRCORE DIAMETER WITH  
 DIFFERENT AXIAL TO TANGENTIAL FLOW RATE  
 RATIOS USING DIFFERENT AIR RATES



TUBE DIAMETER = 3.81 cms  
 TANGENTIAL FLOW RATE = 2240 L.P.H.

SYMBOL	AIR FLOW RATE IN L.P.H.
○	310
△	600
□	960

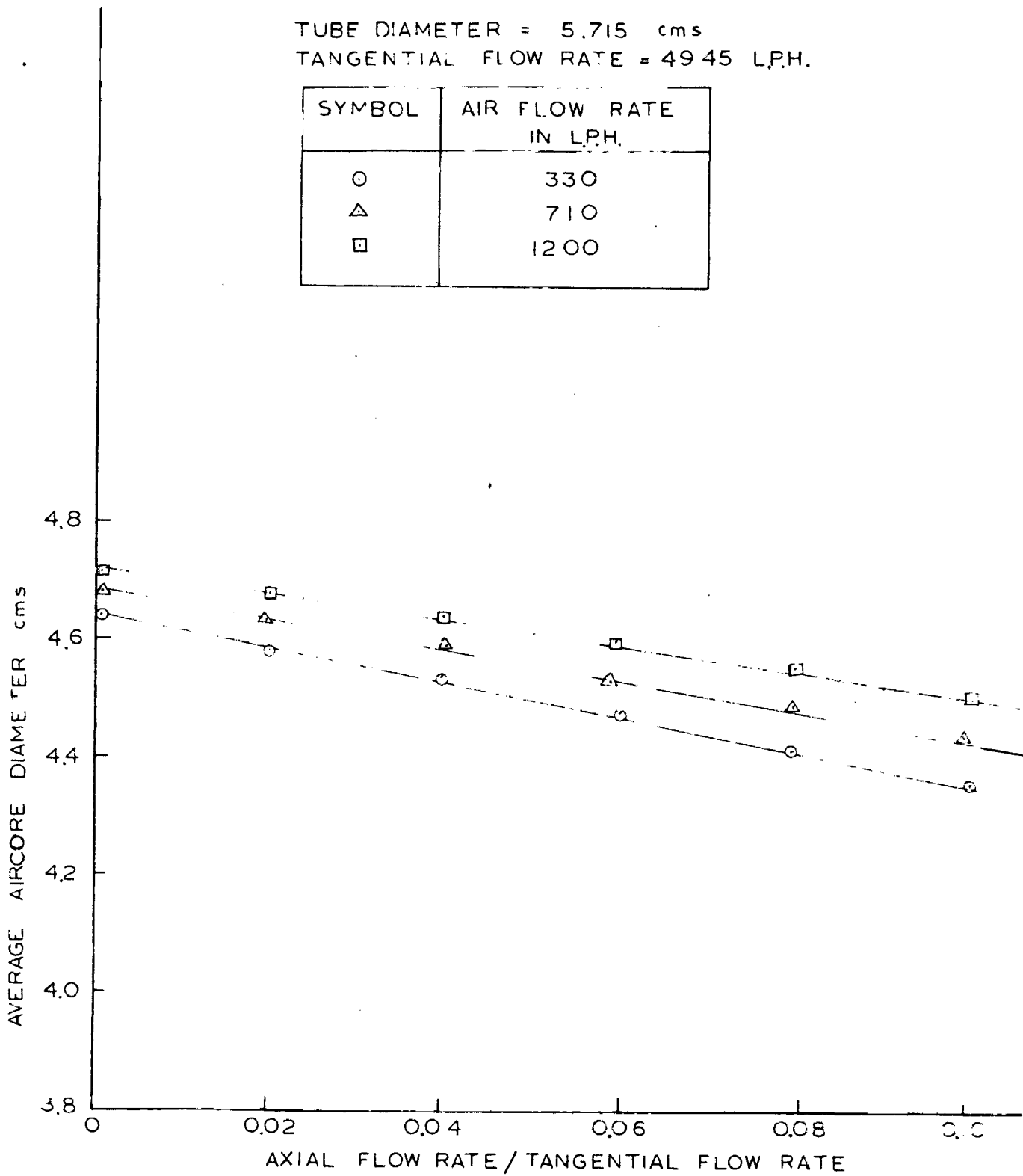


**FIG. 12**

VARIATION OF AVERAGE AIRCORE DIAMETER  
 WITH DIFFERENT AXIAL TO TANGENTIAL FLOW RATE  
 RATIOS USING DIFFERENT AIR RATES

TUBE DIAMETER = 5.715 cms  
TANGENTIAL FLOW RATE = 4945 L.P.H.

SYMBOL	AIR FLOW RATE IN L.P.H.
○	330
△	710
□	1200



**FIG.13**

VARIATION OF AVERAGE AIRCORE DIAMETER WITH  
DIFFERENT AXIAL TO TANGENTIAL FLOW RATE  
RATIOS USING DIFFERENT AIR RATES

TUBE DIAMETER = 5.715 cms

TANGENTIAL FLOW RATE = 3600 L.P.H

SYMBOL	AIR FLOW RATE IN L.P.H
○	330
△	710
□	1200

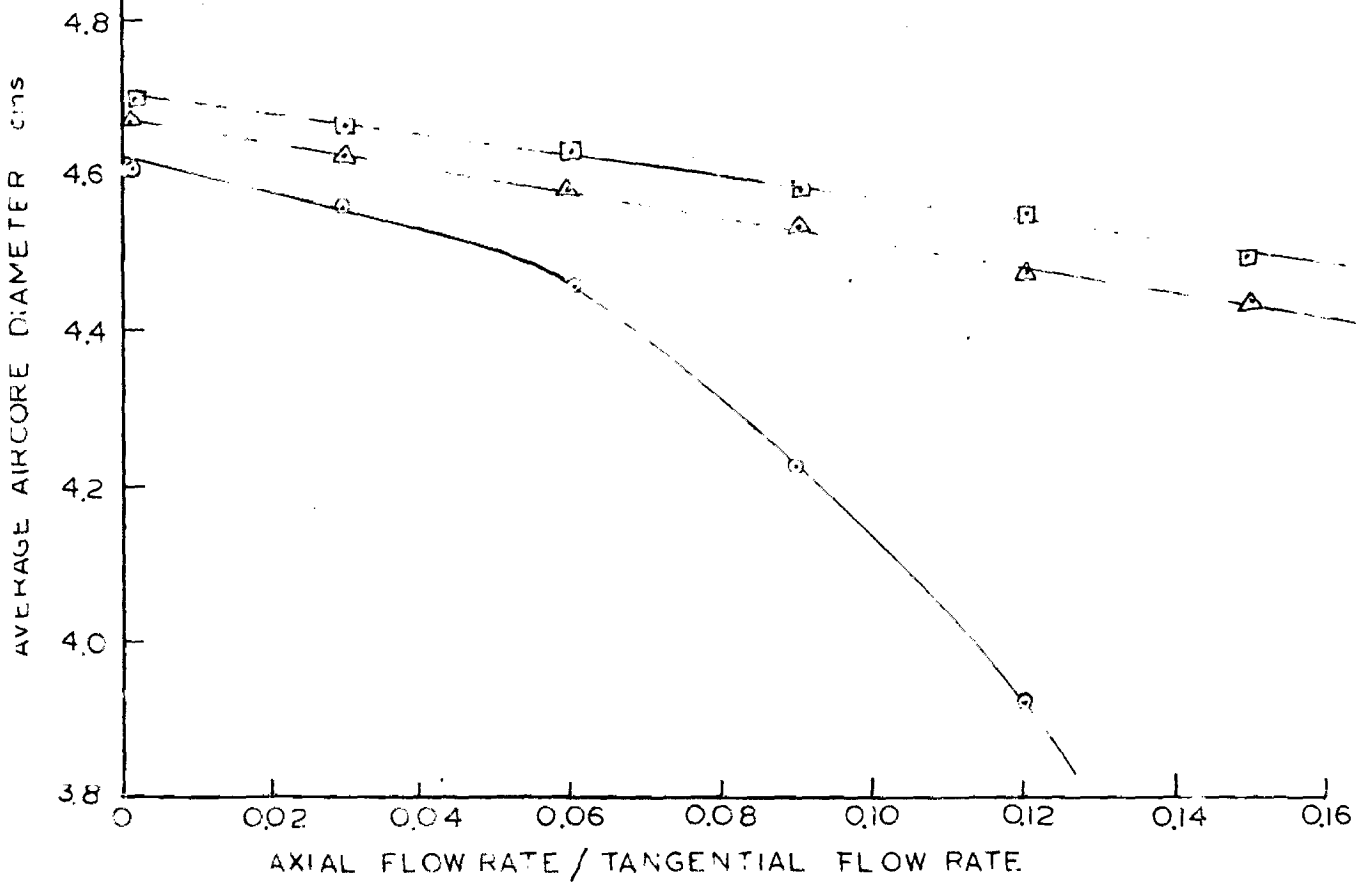
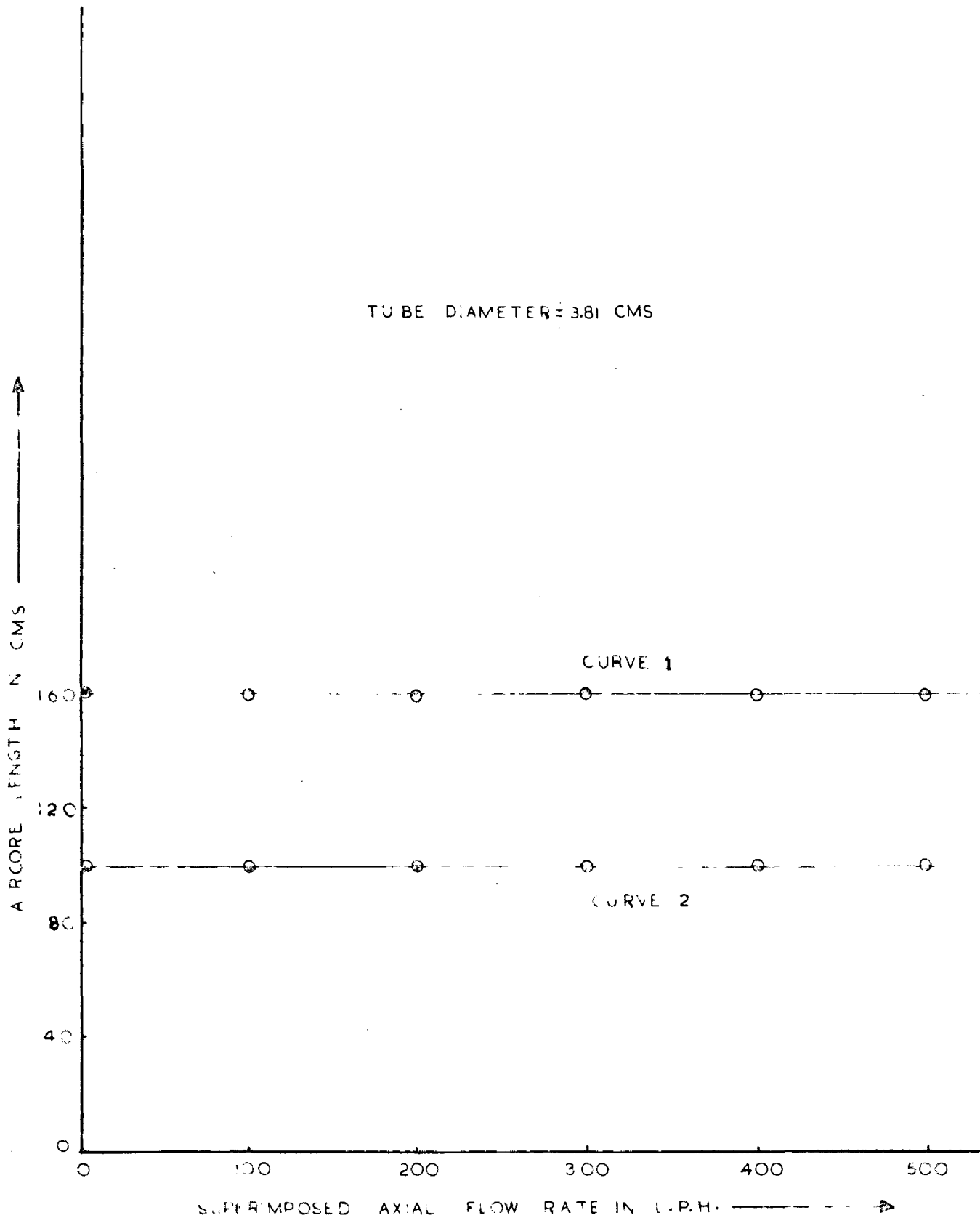


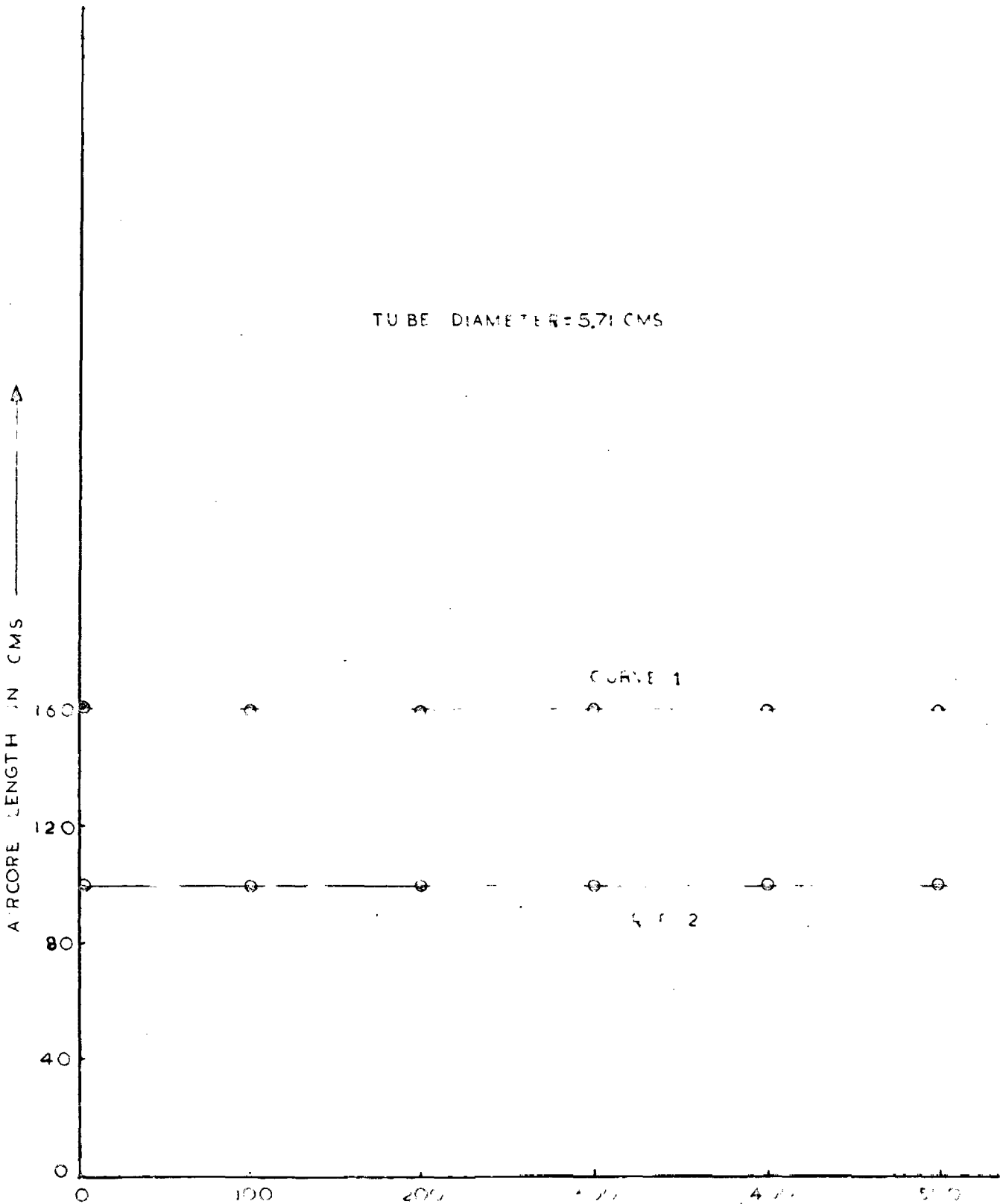
FIG 14

VARIATION OF AVERAGE AIRCORE DIAMETER WITH  
DIFFERENT AXIAL TO TANGENTIAL FLOW RATES  
RATIO USING DIFFERENT AIR RATES



**FIG.15**

VARIATION OF AIRCORE LENGTH WITH DIFFERENT  
SUPERIMPOSED AXIAL FLOW RATES



**FIG.16**

SUPERIMPOSED AXIAL FLOW RATE IN LPM  
 VARIATION OF AIR CORE LENGTH WITH DIFFERENT  
 SUPERIMPOSED AXIAL FLOW RATES

we find that average air core diameter lies in disturbed zone for curve 3 and in stable zone for curve 1 and 2. So we can conclude that, <sup>with</sup> higher air rates more axial flow can be superimposed without disturbing the air core.

Fig. 9 and 10 represent the similar results for the tube of diameter 5.715 cm.

Fig. 11 and 12 for tube diameter of 3.01 cm and 13 & 14 for tube diameter of 5.715 cm, give an idea of the axial to tangential ratios which can be used without disturbing the air core for the given experimental conditions. These figures - also reveals that at low tangential flow rates more axial to tangential ratio can be used without disturbing the air-core by increasing air input rates.

The effect of superimposition of axial flows on the length of the air core is shown in Fig. 15 and 16 for tubes of diameter 3.01 cm and 5.715 cm respectively. It can be seen that the length of the air core is nearly unaffected by the superimposition of axial flows under the experimental conditions of investigation. However, it may be added that some times it was difficult to exactly pin point the length of the air core.

#### RECOMMENDATIONS

From the experimental work carried out it has been found that it is possible to superimpose axial flows on tangential flows without disturbing the air core

provided higher values of air rates are used. By such superimposition of axial flows there shall be less pressure drop for the total flow rate of water in such systems and this will help in reducing the power consumption while using tangential flow systems.

We could not superimpose axial flows more than 500 LPH because of the non availability of rotameter in the flow range of 500 to 1500 LPH. This is why we could not get the critical axial flow rates (i.e. the maximum allowable axial flow which can be superimposed without disturbing the aircore) for higher tangential flow rates and air rates.

Further work should be carried out using a flow meter covering the flow range of 500 to 1500 LPH, to measure a axial flows. It is also desirable that some device may be used to measure more accurately the axial and tangential flow rates, with such an arrangement, critical axial flows can be obtained for different air rates and tangential flow rates. A separate graph can be plotted between critical axial flow rate and tangential flow rate for different air rates to obtain optimum values of air rate and tangential flow rate for a particular tube.

CHAPTER V

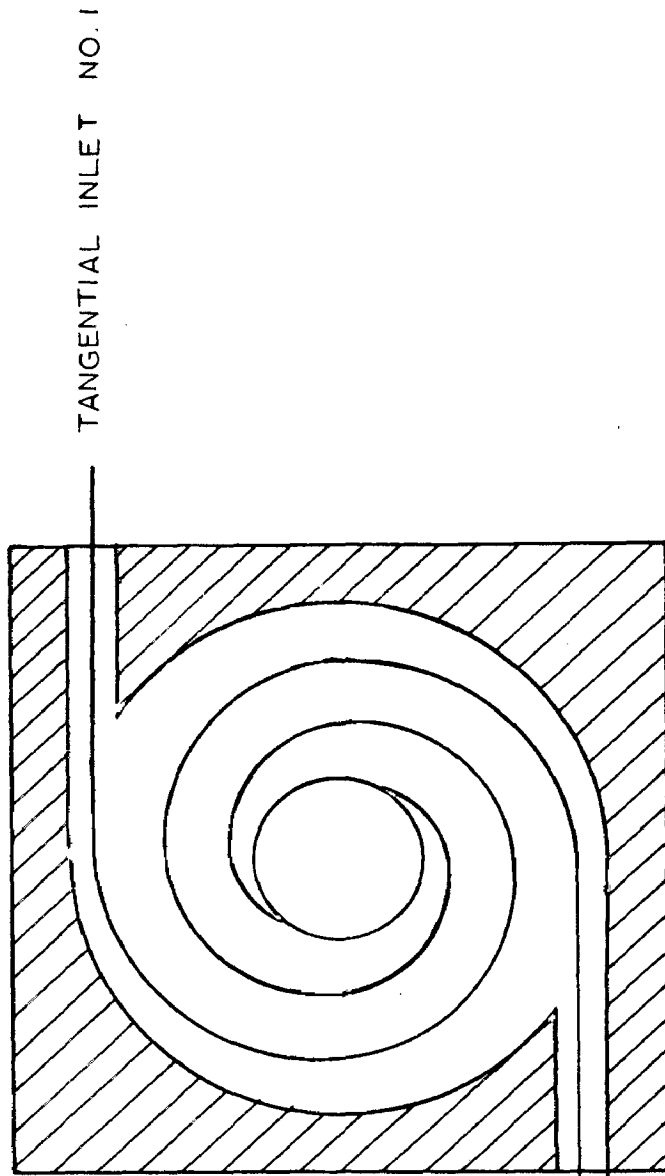
BIBLIOGRAPHY



## BIBLIOGRAPHY

1. Kreith, F. "Advances in Heat Transfer", Vol. 5, Academic press, New York and London (1968)
2. Talbot, L. "Laminar swirling pipe flow," Journal Applied Mechanics, Vol. 21, p.1 (1954)
3. Binnie, A.M. & Teare, J.D. "Experiments on the Flow of Swirling Water through a pressure nozzle and an open Trumpet" Proceedings Royal Society London, Vol. 255 A, p.73 (1956)
4. Binnie, A.M. "Experiments on the slow swirling flow of a viscous liquid through a tube", Quarterly Journal Mech. and Applied Maths, Vol. 10(3), p. 276 (1957)
5. Nuttal J.B. "Axial flow in a vortex" Nature, Vol.172 p. 582(1958).
6. Place, G. "Investigation of Air flow in a spray Drier by Tracer and Model Techniques", Transactions Institution Chemical Engineers, Vol. 57, p. 268 (1959).  
Ridgway, K.  
and Danchenko  
P.V.
7. Kreith, F. and Senyu, O.K. "The decay of turbulent swirl in a pipe", Journal Fluid Mechanics, Vol. 22, p.257(1965)
8. Smithberg, E. and Landos, F. "Friction and forced convection heat transfer characteristics in tubes with twisted tapes swirl generators" Transactions ASME Heat Transfer Journal, Vol. 86, p.39(1964).
9. Brown, V.P. and Hsiao, A.H. "Swirl flow in cylinders" A.I.Ch.E. Journal Vol. 7, p. 583 (1961).

20. King, H.A., "Static Pressure and Velocity profiles in swirling incompressible tube flow". A.I.Ch.E. Journal Vol. 15(6) p. 697(1969).  
Rochester, N.Y. and Kermode, R.L.
11. Lopina R.F. "Heat Transfer and Pressure drop in tape generated swirl flow of single phase water" Transactions ASME Heat Transfer Journal, Vol. 91, p. 434, (1969).  
and Bergles, A.E.
12. Bergles, A.E., "Heat Transfer in rough tubes with tape generated swirl flow" Transaction ASME, Heat Transfer Journal Vol. 91, p. 445(1969).  
Lee, R.A. & Nistic, B.B.
13. Sharma, D.D. Ph.D. Thesis on "Studies on the Momentum Transfer and Stability of Swirling Flow in Long Horizontal Tubes" University of Roorkee, Roorkee.
14. Sharma, D.D., "Studies on the stability of swirling flows in long tubes" Indian Journal of Technology Vol. 10, p.130 (1972).  
Panesar, P.S. and Gopal Mishra, H.
15. Chandra Y. M.B. Thesis on "Studies on Downward Swirling flow in vertical Pipes". University of Roorkee, Roorkee, 1974.
16. Gopal C.R.S. M.B. Thesis on "Studies on Upward Swirling flows in long tubes" University of Roorkee Roorkee, 1974.



TANGENTIAL INLET  
NO 2

TANGENTIAL INLET NO. 1



PRODUCTION OF SWIRLING FLOW BY  
TANGENTIAL INJECTION OF LIQUID